

MENG366

Mathematical Modeling of Mechanical and Electrical Systems & Linearization

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Translational Mechanical Systems

- Basic (Idealized) Modeling Elements
- Interconnection Relationships -Physical Laws
- Derive Equation of Motion (EOM) - SDOF
- Energy Transfer
- Series and Parallel Connections
- Derive Equation of Motion (EOM) - MDOF

Key Concepts to Remember

- Three primary elements of interest
 - Mass (inertia) M
 - Stiffness (spring) K
 - Dissipation (damper) B
 - Usually we deal with “equivalent” M, K, B
 - *Distributed mass* \rightarrow *lumped mass*
- Lumped parameters
 - Mass maintains motion (Kinetic Energy)
 - Stiffness restores motion (Potential Energy)
 - Damping eliminates motion ~~(Eliminate Energy ?)~~
(Absorb Energy)

Variables

- x : *displacement* [m]
- v : *velocity* [m/sec]
- a : *acceleration* [m/sec²]
- f : *force* [N]
- p : *power* [Nm/sec]
- w : *work* (energy) [Nm]
1 [Nm] = 1 [J] (Joule)

$$v = \dot{x} = \frac{d}{dt} x$$

$$a = \frac{dv}{dt} = \frac{d}{dt} \left(\frac{d}{dt} x \right) = \frac{d^2}{dt^2} x = \ddot{x}$$

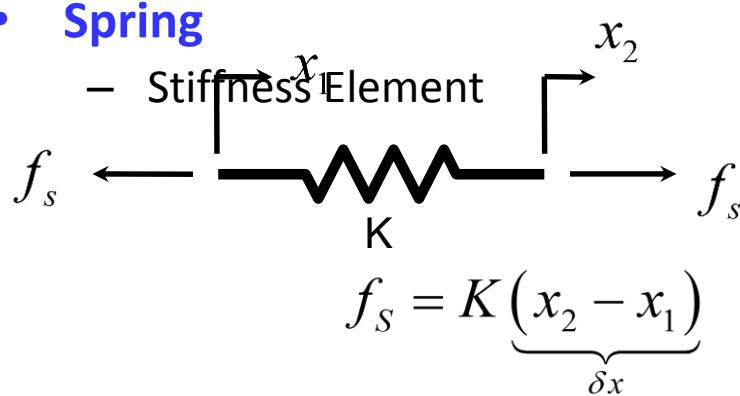
$$p = f \cdot v = f \cdot \dot{x} = \frac{d}{dt} w$$

$$w(t_1) = w(t_0) + \int_{t_0}^{t_1} p(t) dt$$

$$= w(t_0) + \int_{t_0}^{t_1} (f \cdot \dot{x}) dt$$

Basic (Idealized) Modeling Elements

- **Spring**



- Idealization

- Massless
- No Damping
- Linear

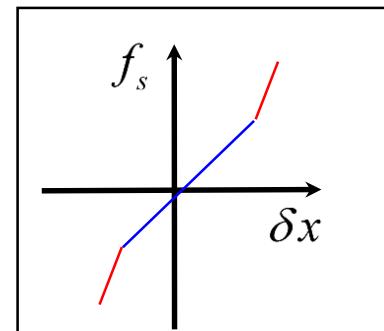
- Stores Energy

Potential Energy

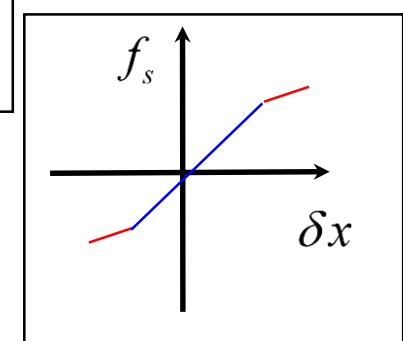
$$U = \frac{1}{2} k (\delta x)^2$$

- Reality

- 1/3 of the spring mass may be considered into the lumped model.
- In large displacement operation springs are *nonlinear*.



Hard Spring



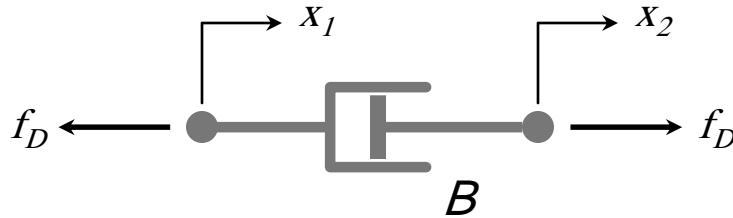
Soft Spring

Linear spring
 \rightarrow *nonlinear spring*
 \rightarrow *broken spring !!*

Basic Modeling Elements

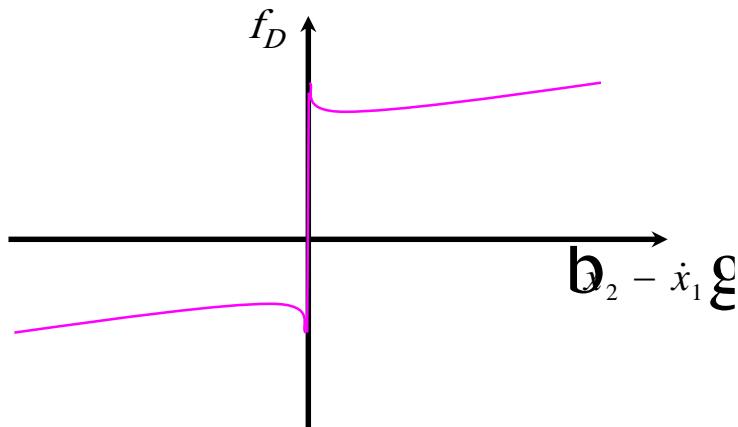
- **Damper**

- Friction Element



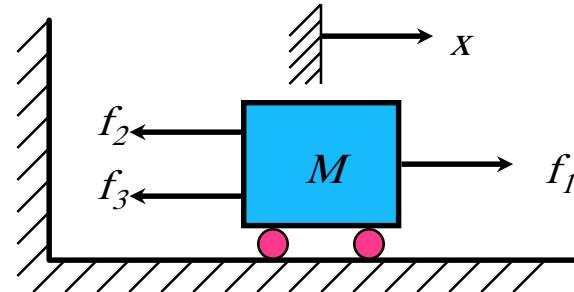
$$f_D = B \mathbf{b}_2 - \dot{x}_1 \mathbf{g} = B \mathbf{b}_2 - v_1 \boldsymbol{\xi}$$

- Dissipate Energy



- **Mass**

- Inertia Element



$$M \ddot{x} = \sum_i f_i = f_1 - f_2 - f_3$$

- Stores Kinetic Energy

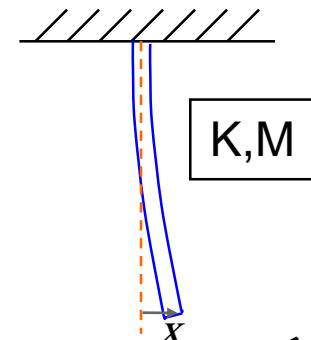
$$T = \frac{1}{2} M \dot{x}^2$$

Interconnection Laws

- Newton's Second Law

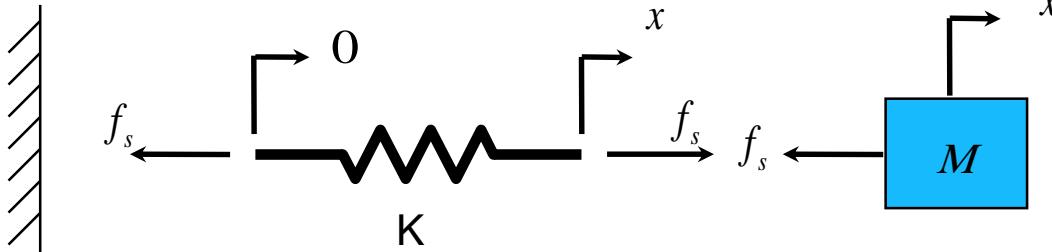
$$\frac{d}{dt} \underbrace{\mathbf{b}_M v g}_{\text{Linear Momentum}} = M \ddot{x} = \sum_i f_{EXTi}$$

Lumped Model of a Flexible Beam



- Newton's Third Law

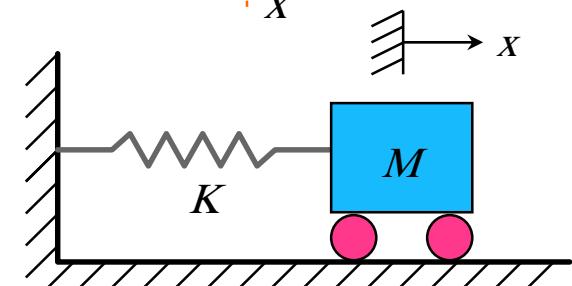
- Action & Reaction Forces



$$f_s = K(x - 0) = Kx$$

Massless spring

$$\begin{aligned} M \ddot{x} &= -f_s \\ M \ddot{x} &= -Kx \\ M \ddot{x} + Kx &= 0 \end{aligned}$$



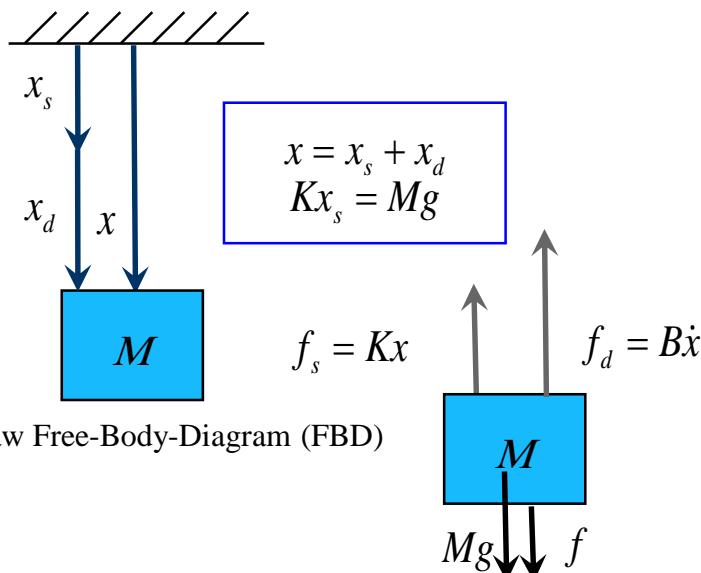
E.O.M.

Modeling Steps

- *Understand System Function, Define Problem, and Identify Input/Output Variables*
- *Draw Simplified Schematics Using Basic Elements*
- *Develop Mathematical Model (Diff. Eq.)*
 - Identify reference point and positive direction.
 - Draw Free-Body-Diagram (FBD) for each basic element.
 - Write Elemental Equations as well as Interconnecting Equations by applying physical laws. (*Check: # eq = # unk*)
 - Combine Equations by eliminating intermediate variables.
- *Validate Model by Comparing Simulation Results with Physical Measurements*

Single Degree of Freedom (SDOF) System

- Define Problem *The motion of the object*
- Input f
- Output x
- Develop Mathematical Model (Diff. Eq.)
 - Identify reference point and positive direction.



- Draw Free-Body-Diagram (FBD)
- Write Elemental Equations

$$M\ddot{x} = -B\dot{x} - Kx + Mg + f$$

$$M\ddot{x} + B\dot{x} + Kx = Mg + f$$

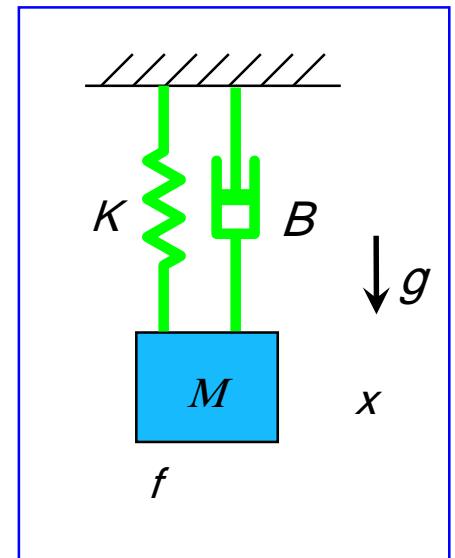
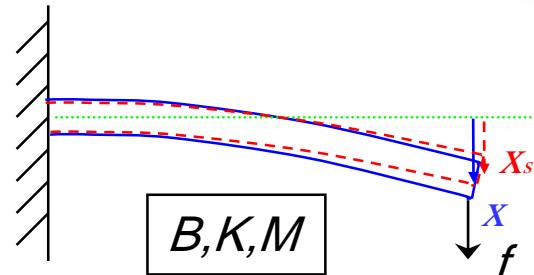
$$M\ddot{x}_u + B\dot{x}_u + Kx_u = Mg + f$$

From the undeformed position

$$M\ddot{x}_d + B\dot{x}_d + Kx_d = f$$

From the deformed (static equilibrium) position

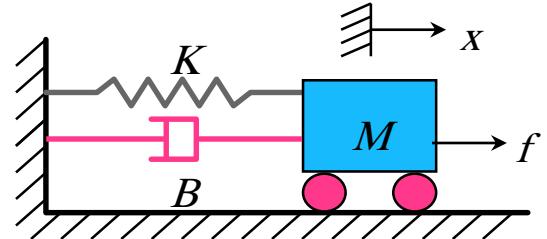
- Validate Model by Comparing Simulation Results with Physical Measurement



Energy Distribution

- EOM of a simple Mass-Spring-Damper System**

$$\underbrace{M\ddot{x}}_{\text{Contribution of Inertia}} + \underbrace{B\dot{x}}_{\text{Contribution of the Damper}} + \underbrace{Kx}_{\text{Contribution of the Spring}} = \underbrace{f(t)}_{\text{Total Applied Force}}$$



We want to look at the energy distribution of the system. How should we start ?

- Multiply the above equation by the velocity term v :** \Leftarrow What have we done ?

$$M\ddot{x} \cdot \dot{x} + B\dot{x} \cdot \dot{x} + Kx \cdot \dot{x} = f(t) \cdot \dot{x}$$

- Integrate the second equation w.r.t. time:** \Leftarrow What are we doing now ?

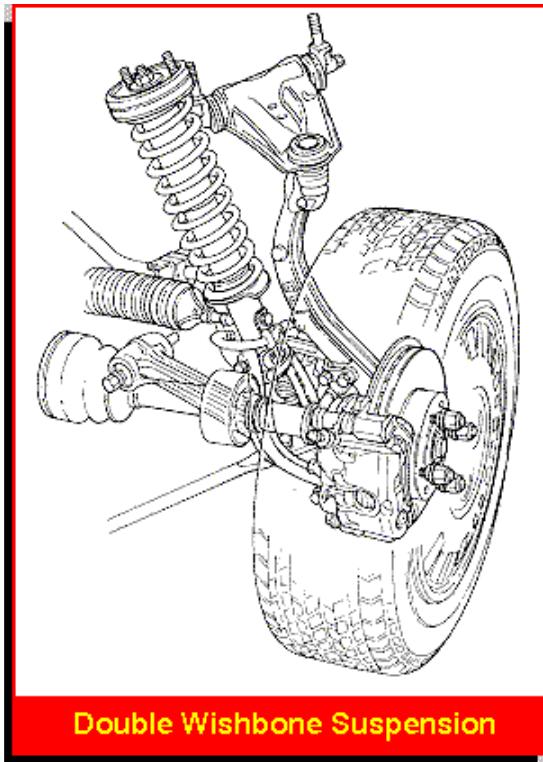
$$\underbrace{\int_{t_0}^{t_1} M\ddot{x} \cdot \dot{x} dt}_{\Delta KE} + \underbrace{\int_{t_0}^{t_1} B\dot{x} \cdot \dot{x} dt}_{\int_{t_0}^{t_1} B\dot{x}^2 dt \geq 0} + \underbrace{\int_{t_0}^{t_1} Kx \cdot \dot{x} dt}_{\Delta PE} = \underbrace{\int_{t_0}^{t_1} f(t) \cdot \dot{x} dt}_W$$

\Downarrow Change of kinetic energy \Downarrow Energy dissipated by damper \Downarrow Change of potential energy
 Total work done by the applied force $f(t)$ from time t_0 to t_1

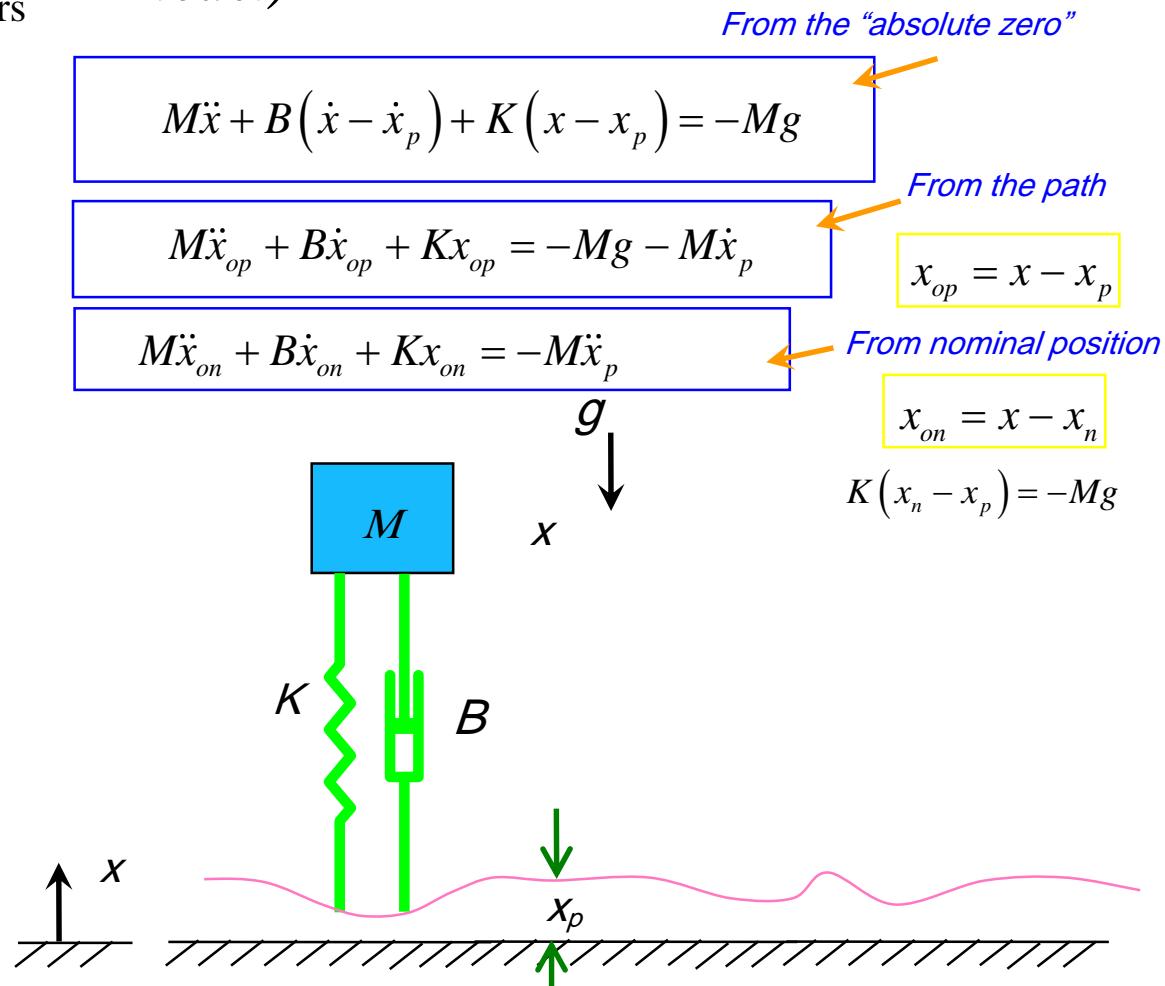
SDOF Suspension (Example)

- Suspension System**

Minimize the effect of the surface roughness of the road on the drivers comfort.

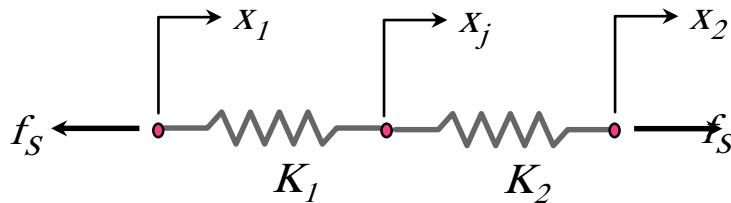
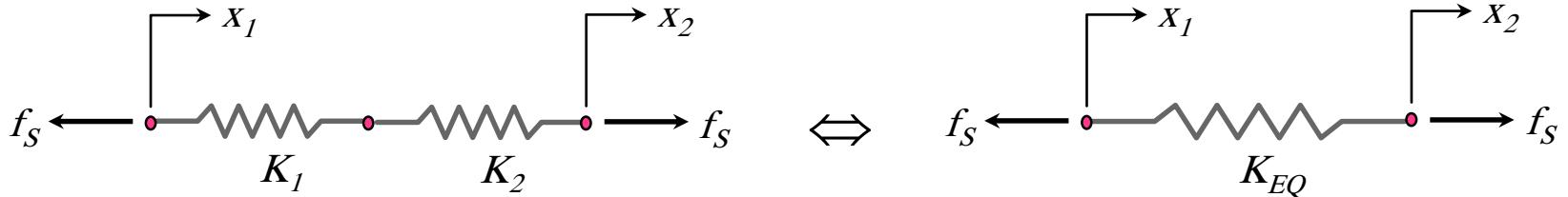


- Simplified Schematic (neglecting tire model)*



Series Connection

- Springs in Series



$$K_1(x_j - x_1) = K_2(x_2 - x_j)$$

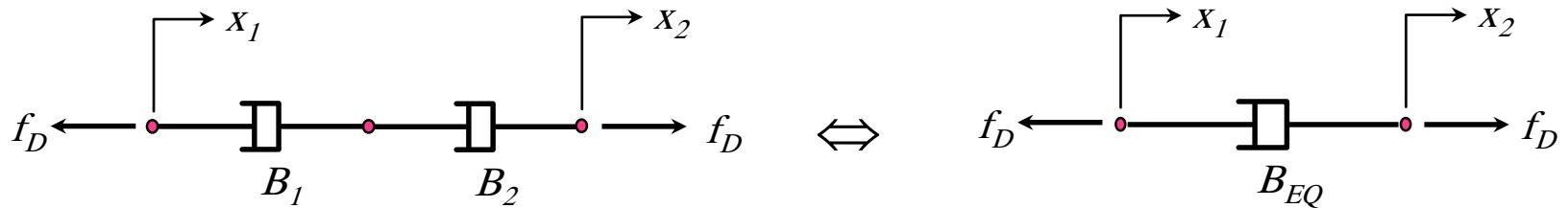
$$x_j = \frac{1}{K_1 + K_2} [K_2 x_2 + K_1 x_1]$$

$$f_s = K_1(x_j - x_1) = K_1 \left\{ \underbrace{\frac{1}{K_1 + K_2} [K_2 x_2 + K_1 x_1]}_{x_j} - x_1 \right\}$$

$$f_s = \frac{K_1 K_2}{\underbrace{K_1 + K_2}_{K_{eq}}} [x_2 - x_1]$$

Series Connection

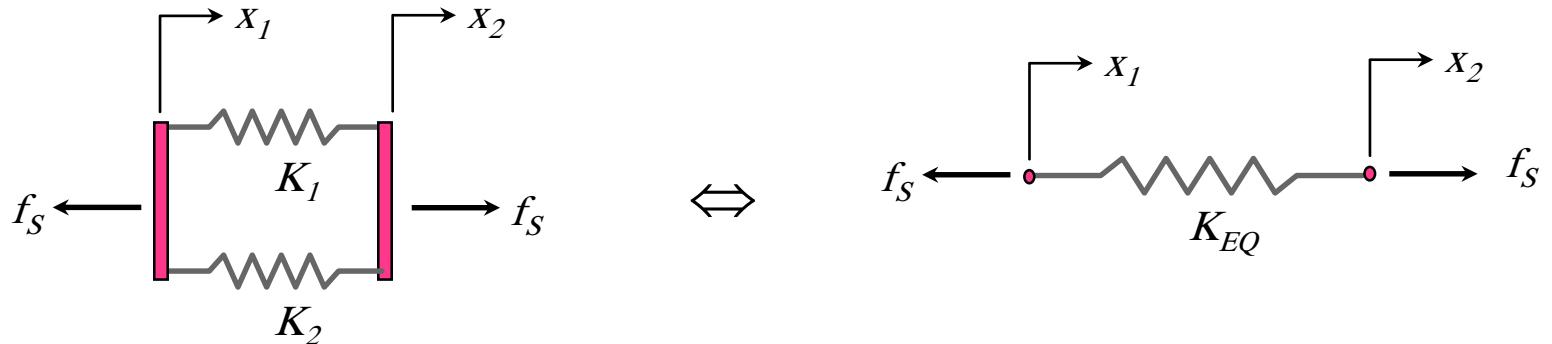
- Dampers in Series



$$f_d = \underbrace{\frac{B_1 B_2}{B_1 + B_2}}_{B_{eq}} [\dot{x}_2 - \dot{x}_1]$$

Parallel Connection

- Springs in Parallel

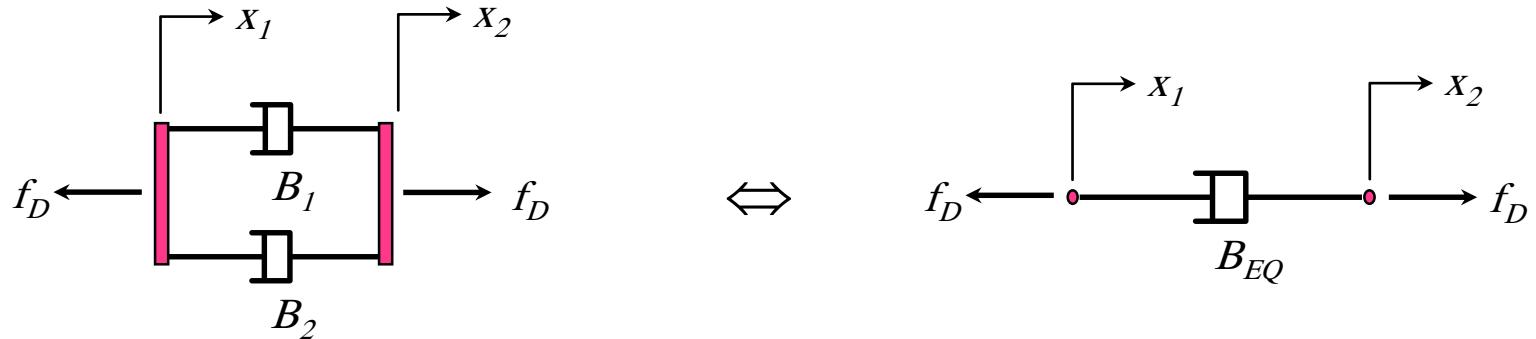


$$f_s = K_1(x_2 - x_1) + K_2(x_2 - x_1)$$

$$f_s = \underbrace{(K_1 + K_2)}_{K_{eq}}(x_2 - x_1)$$

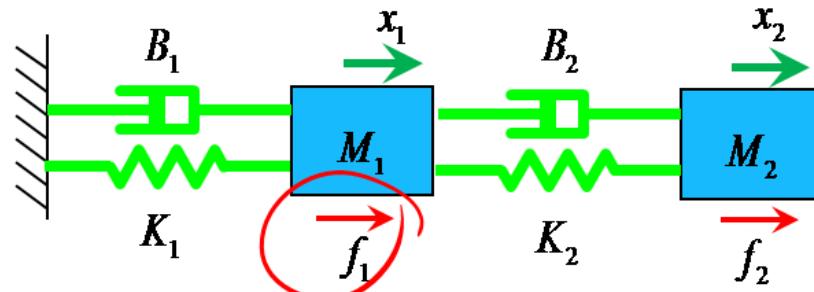
Parallel Connection

Dampers in Parallel

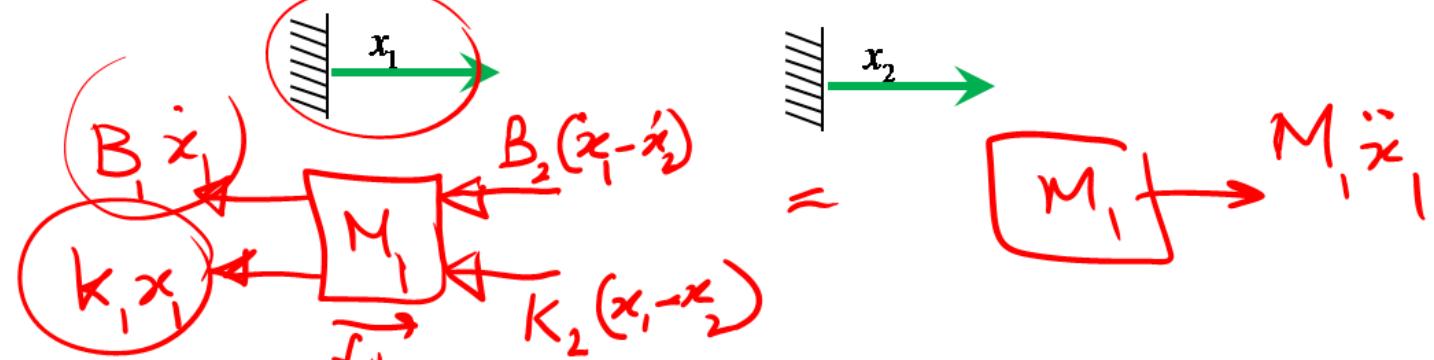


$$f_s = \underbrace{(B_1 + B_2)}_{K_{eq}} (\dot{x}_2 - \dot{x}_1)$$

Two Degree of Freedom (TDOF) System



For x_1 :



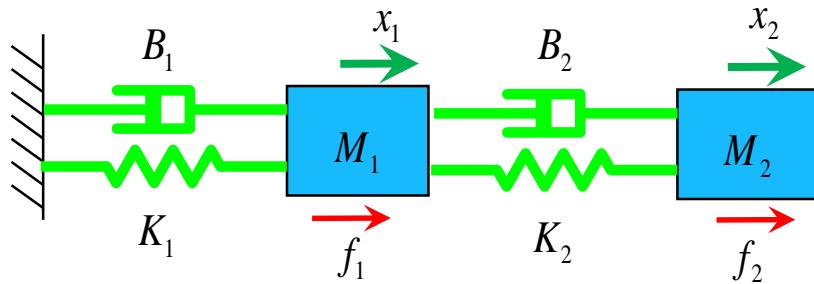
$$-B_1 \dot{x}_1 - k_1 x_1 - B_2(\dot{x}_1 - \dot{x}_2) - K_2(x_1 - x_2) + f_1 = M_1 \ddot{x}_1$$

$$-B_1 \dot{x}_1 - k_1 x_1 - B_2 \dot{x}_2 + B_2 x_2 - K_2 x_1 + k_2 x_2 + f_1 = M_1 \ddot{x}_1$$

$$M_1 \ddot{x}_1 + (B_1 + B_2) \dot{x}_1 + (k_1 + k_2) x_1 - B_2 \dot{x}_2 - K_2 x_2 = f_1$$

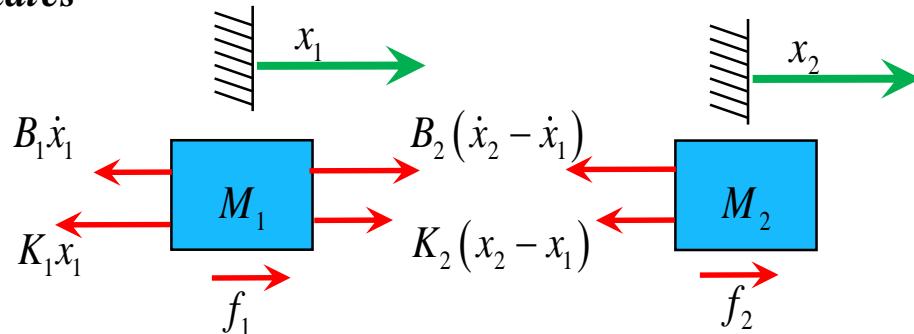
Two Degree of Freedom (TDOF) System

- $DOF = 2$



- *Absolute coordinates*

- *FBD*

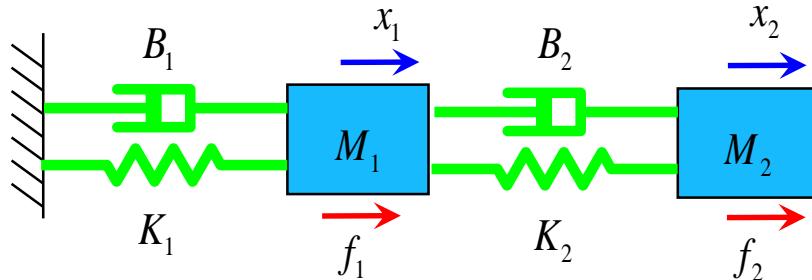


- *Newton's law*

$$M_1 \ddot{x}_1 = -B_1 \dot{x}_1 - K_1 x_1 + B_2 (\dot{x}_2 - \dot{x}_1) + K_2 (x_2 - x_1) + f_1(t)$$

$$M_2 \ddot{x}_2 = -B_2 (\dot{x}_2 - \dot{x}_1) - K_2 (x_2 - x_1) + f_2(t)$$

Two Degree of Freedom (TDOF) System



Static coupling

- Absolute coordinates*

$$M_1 \ddot{x}_1 + (B_1 + B_2) \dot{x}_1 + (K_1 + K_2) x_1 - B_2 \dot{x}_2 - K_2 x_2 = f_1(t)$$

$$M_2 \ddot{x}_2 + B_2 \dot{x}_2 + K_2 x_2 - B_2 \dot{x}_1 - K_2 x_1 = f_2(t)$$

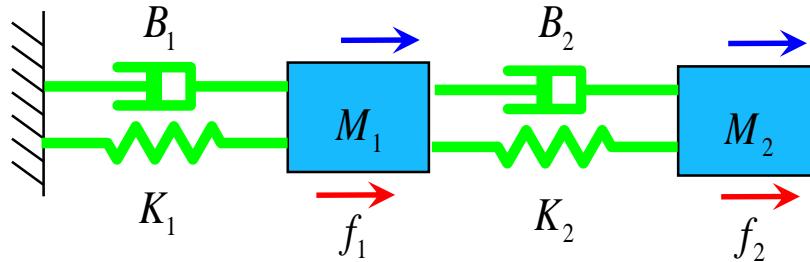
- Relative coordinates*

$$M_1 \ddot{x}_1 + B_1 \dot{x}_1 + K_1 x_1 - B_2 \dot{x}_{21} - K_2 x_{21} = f_1(t)$$

$$M_2 \ddot{x}_1 + M_2 \ddot{x}_{21} + B_2 \dot{x}_{21} + K_2 x_{21} = f_2(t)$$

Dynamic coupling

Two DOF System – Matrix Form of EOM



- Absolute coordinates*

Mass matrix
Damping matrix
Output vector

$$\begin{bmatrix} M_1 & 0 \\ 0 & M_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} B_1 + B_2 & -B_2 \\ -B_2 & B_2 \end{bmatrix} \begin{Bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{Bmatrix} + \begin{bmatrix} K_1 + K_2 & -K_2 \\ -K_2 & K_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} f_1(t) \\ f_2(t) \end{Bmatrix}$$

- Relative coordinates*

Stiffness matrix SYMMETRIC

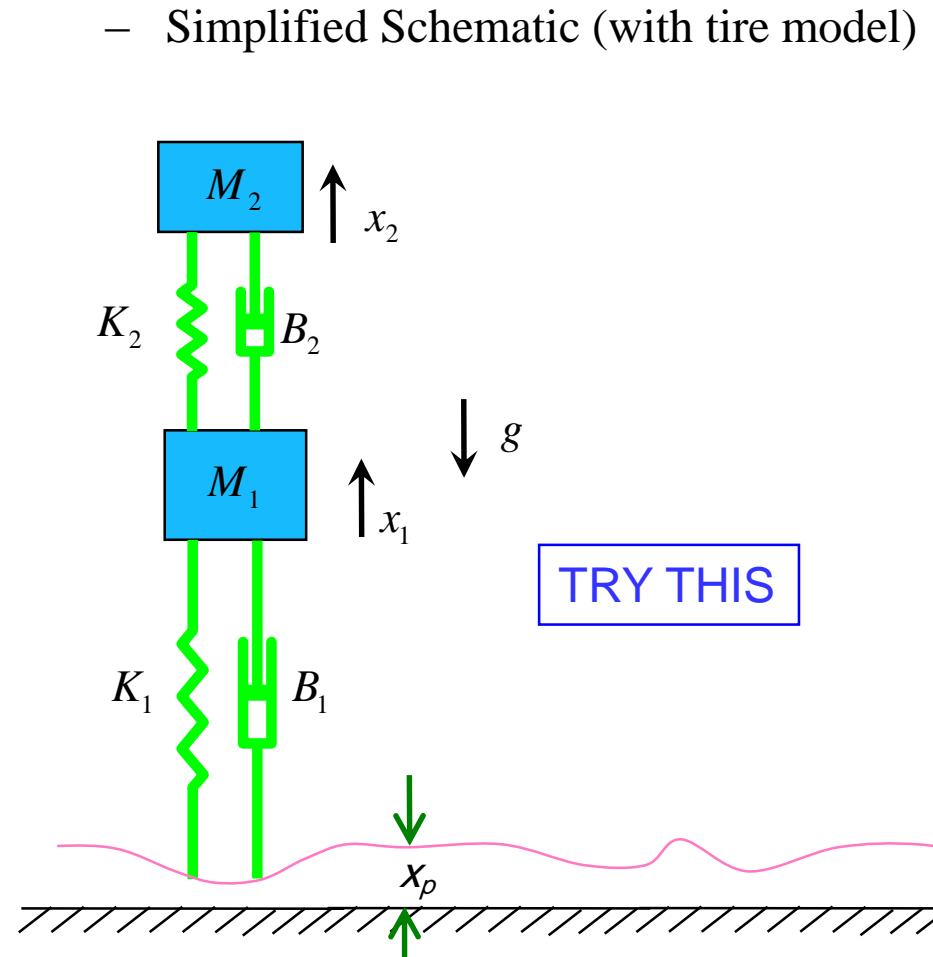
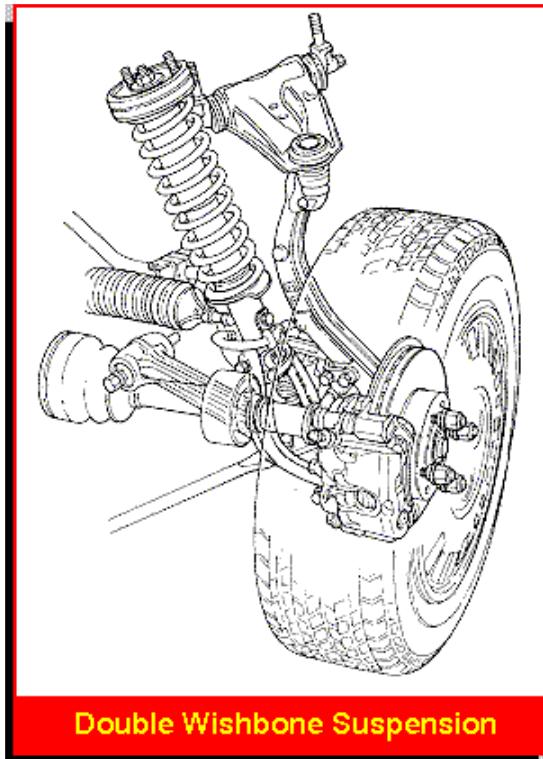
$$\begin{bmatrix} M_1 & 0 \\ M_2 & M_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_{21} \end{Bmatrix} + \begin{bmatrix} B_1 & -B_2 \\ 0 & B_2 \end{bmatrix} \begin{Bmatrix} \dot{x}_1 \\ \dot{x}_{21} \end{Bmatrix} + \begin{bmatrix} K_1 & -K_2 \\ 0 & K_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_{21} \end{Bmatrix} = \begin{Bmatrix} f_1(t) \\ f_2(t) \end{Bmatrix}$$

NON-SYMMETRIC

MDOF Suspension

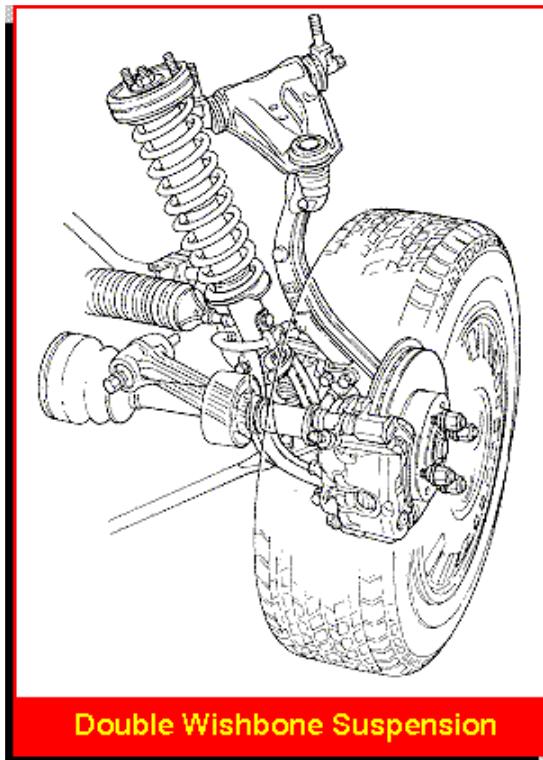
Example 1

• *Suspension System*

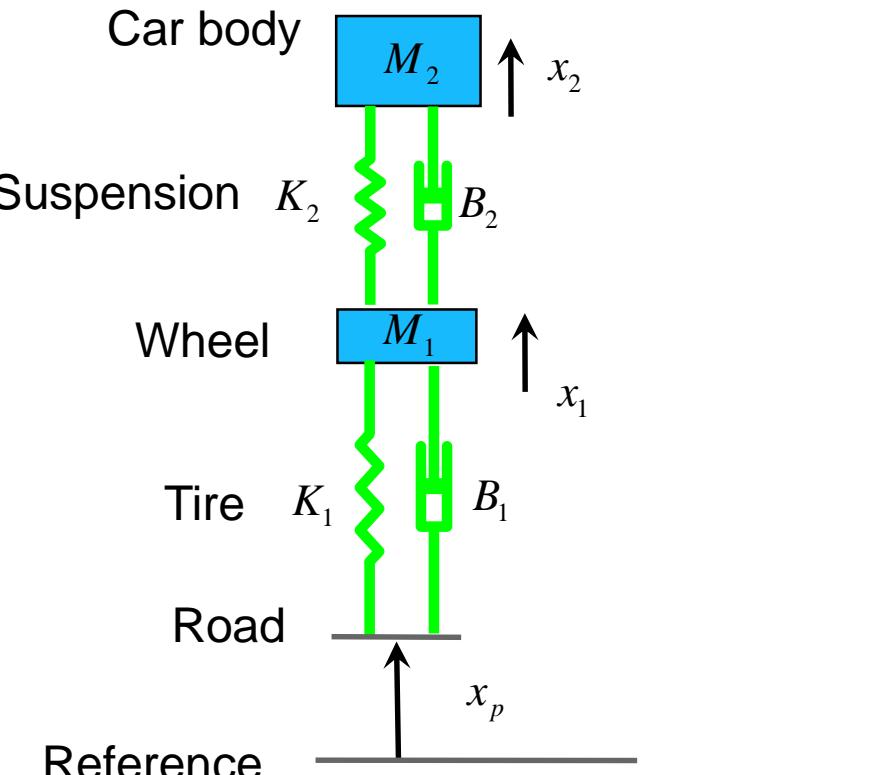


MDOF Suspension

- Suspension System**

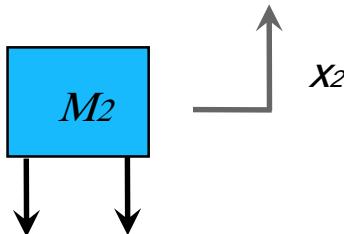


- Simplified Schematic (with tire model)
 - Assume ref. is when springs are Deflected by weights

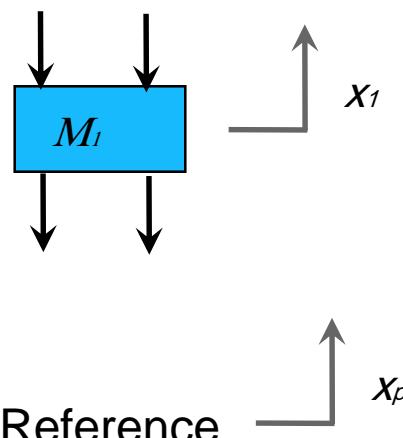


MDOF Suspension

– Draw FBD



– Apply Newton's 2nd Laws



$$\Rightarrow M_2 \ddot{x}_2 + B_2 \dot{x}_2 - B_2 \dot{x}_1 + K_2 x_2 - K_2 x_1 = 0$$

$$\Rightarrow M_1 \ddot{x}_1 - B_2 \dot{x}_2 + (B_2 + B_1) \dot{x}_1 - K_2 x_2 + (K_2 + K_1) x_1 = B_1 \dot{x}_p + K_1 x_p$$

MDOF Suspension

- *Matrix Form*

$$M_2 \ddot{x}_2 + B_2 \dot{x}_2 - B_2 \dot{x}_1 + K_2 x_2 - K_2 x_1 = 0$$

$$M_1 \ddot{x}_1 - B_2 \dot{x}_2 + (B_2 + B_1) \dot{x}_1 - K_2 x_2 + (K_2 + K_1) x_1 = B_1 \dot{x}_p + K_1 x_p$$

Define vector $x = \begin{Bmatrix} x_2 \\ x_1 \end{Bmatrix}$

$$\begin{bmatrix} M_2 & 0 \\ 0 & M_1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_2 \\ \ddot{x}_1 \end{Bmatrix} + \begin{bmatrix} B_2 & -B_2 \\ -B_2 & B_2 + B_1 \end{bmatrix} \begin{Bmatrix} \dot{x}_2 \\ \dot{x}_1 \end{Bmatrix} + \begin{bmatrix} K_2 & -K_2 \\ -K_2 & K_2 + K_1 \end{bmatrix} \begin{Bmatrix} x_2 \\ x_1 \end{Bmatrix} = \begin{Bmatrix} 0 \\ B_1 \dot{x}_p + K_1 x_p \end{Bmatrix}$$

Mass matrix

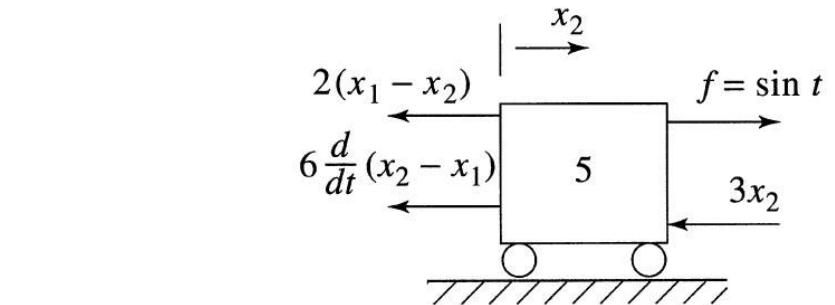
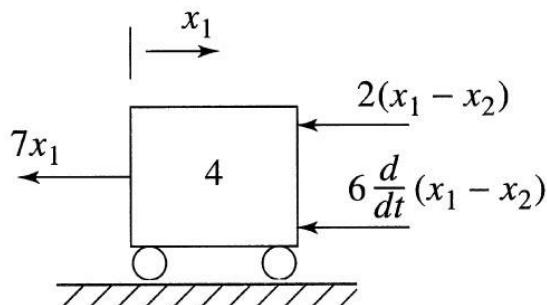
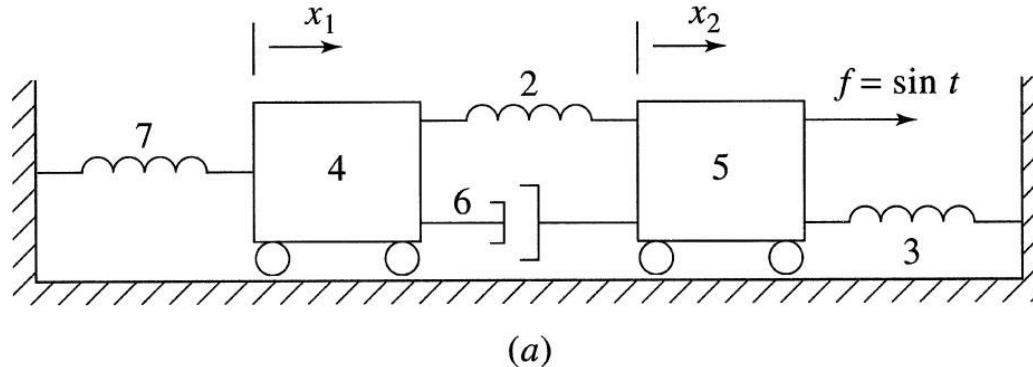
Damping matrix

Stiffness matrix

Input Vector

Two Degree of Freedom (TDOF) System

POP Quiz



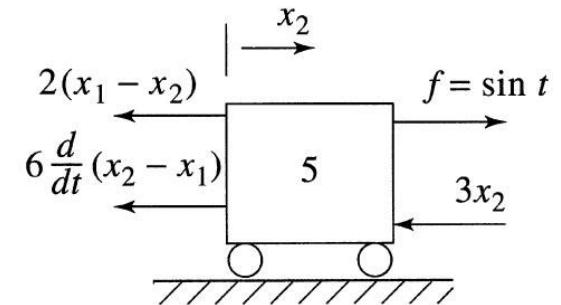
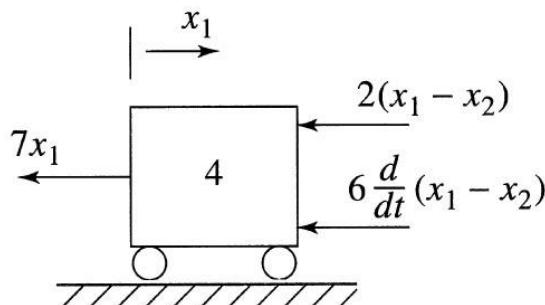
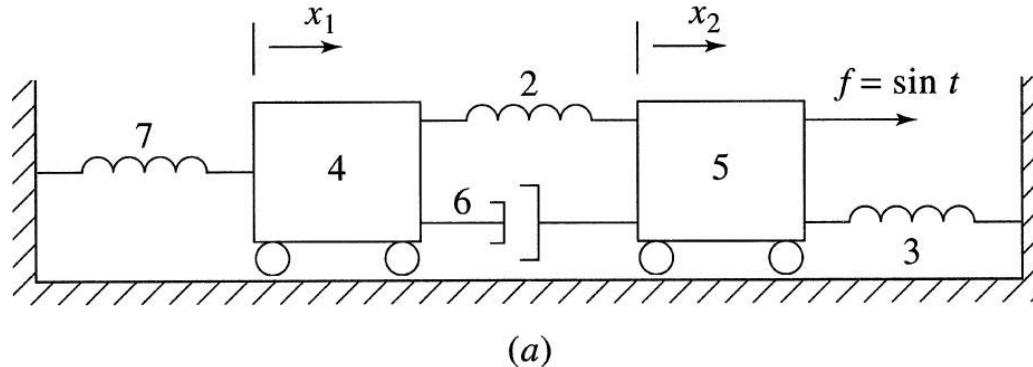
(b)

$$7x_1 + 4 \frac{d^2 x_1(t)}{dt^2} + 2(x_1 - x_2) + 6 \frac{d(x_1 - x_2)}{dt} = 0$$

$$2(x_2 - x_1) + 6 \frac{d(x_2 - x_1)}{dt} + 5 \frac{d^2 x_2}{dt^2} + 3x_2 = \sin t$$

Two Degree of Freedom (TDOF) System

Solution:



(b)

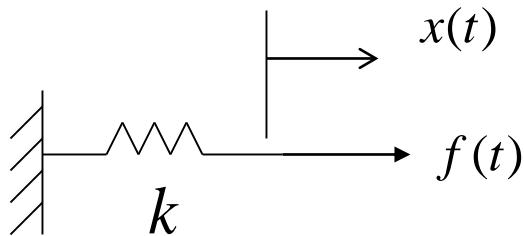
$$7x_1 + 4 \frac{d^2 x_1(t)}{dt^2} + 2(x_1 - x_2) + 6 \frac{d(x_1 - x_2)}{dt} = 0$$

$$2(x_2 - x_1) + 6 \frac{d(x_2 - x_1)}{dt} + 5 \frac{d^2 x_2}{dt^2} + 3x_2 = \sin t$$

Rotational Mechanical Systems

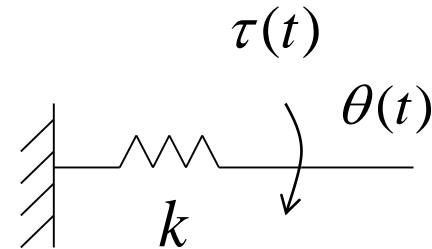
Translational mechanical components

spring



$$f(t) = kx(t)$$

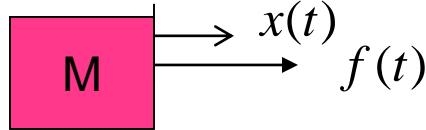
Rotational mechanical components



$$\tau(t) = k\theta(t)$$

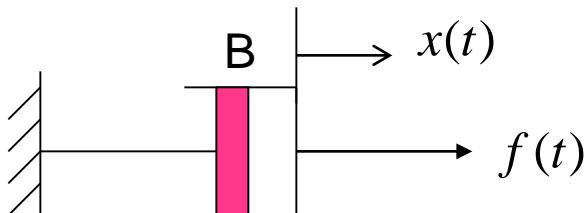
Rotational Mechanical Systems

Translational mechanical components



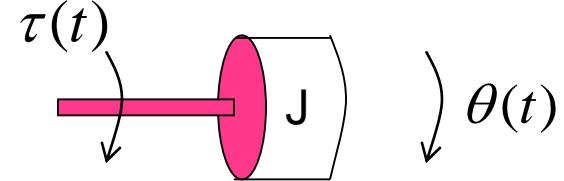
$$f(t) = Ma = Mx''(t)$$

Viscous friction (linear)

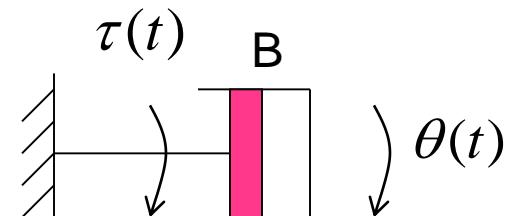


$$f(t) = Bv(t) = Bx'(t)$$

Rotational mechanical components



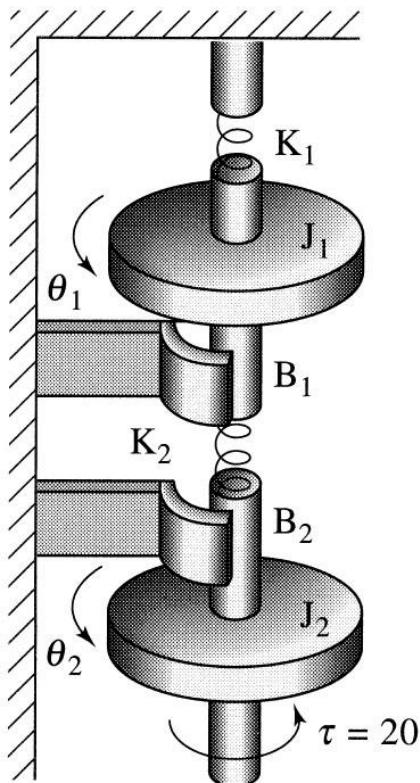
$$\tau(t) = J\theta''(t)$$



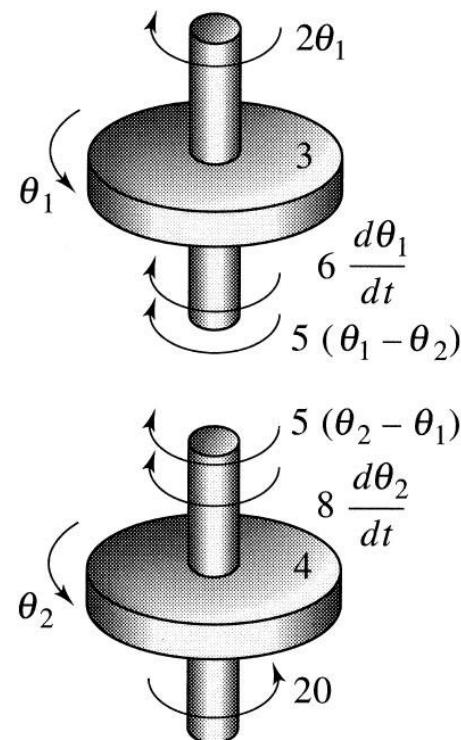
$$\tau(t) = B\theta'(t)$$

Mechanical Systems

Example 2



(a)



(b)

$$\left\{ \begin{array}{l} 3 \frac{d^2\theta_1}{dt^2} + 2\theta_1 + 5(\theta_1 - \theta_2) + 6 \frac{d\theta_1}{dt} = 0 \\ 4 \frac{d^2\theta_2}{dt^2} + 5(\theta_2 - \theta_1) + 8 \frac{d\theta_2}{dt} = 20 \end{array} \right.$$

Mechanical Systems

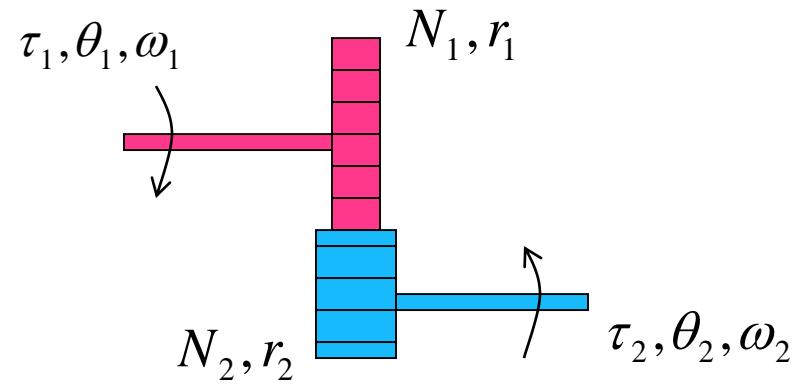
Gear train

$$(1) \quad \frac{r_1}{r_2} = \frac{N_1}{N_2}$$

$$\therefore r \propto N$$

$$(4) \quad r_1 \omega_1 = r_2 \omega_2$$

$$\therefore S_1 = S_2$$



$$(2) \quad r_1 \theta_1 = r_2 \theta_2$$

$$\therefore S_1 = S_2$$

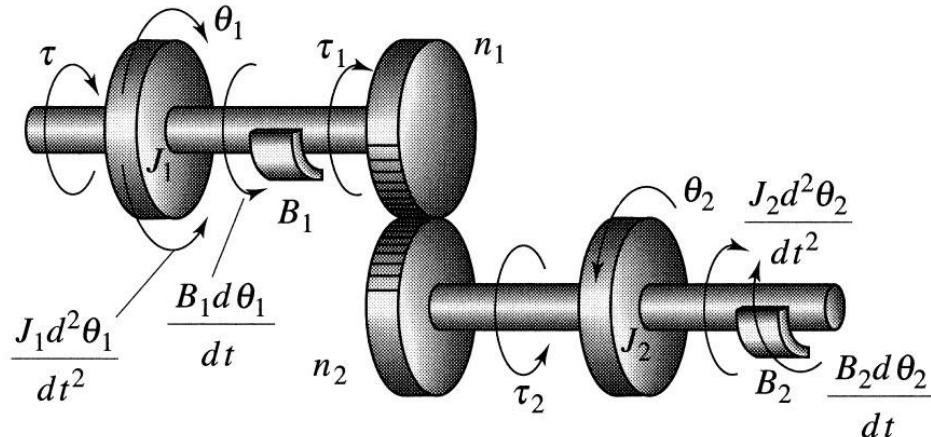
$$(3) \quad \tau_1 \theta_1 = \tau_2 \theta_2$$

no energy loss

$$\frac{N_1}{N_2} = \frac{r_1}{r_2} = \frac{\tau_1}{\tau_2} = \frac{\theta_2}{\theta_1} = \frac{\omega_2}{\omega_1}$$

Mechanical Systems

Example 3



(a)

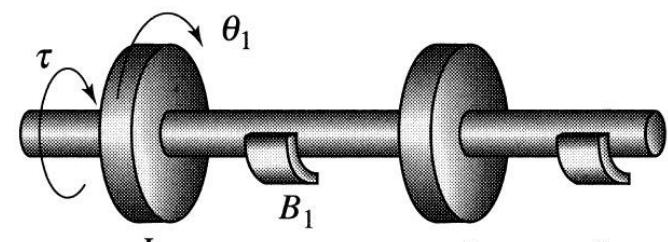
$$\tau = J_1 \ddot{\theta}_1 + B_1 \dot{\theta}_1 + \tau_1 \dots \quad (1)$$

$$\tau_2 = J_2 \ddot{\theta}_2 + B_2 \dot{\theta}_2 \dots \quad (2)$$

$$\tau_2 \Rightarrow \frac{N_2}{N_1} \tau_1$$

$$\ddot{\theta}_2 \Rightarrow \frac{N_1}{N_2} \ddot{\theta}_1$$

$$\dot{\theta}_2 \Rightarrow \frac{N_1}{N_2} \dot{\theta}_1$$



(b)

Electrical Systems

- ***Basic Modeling Elements***
- ***Interconnection Relationships***
- ***Derive Input/Output Models***

Variables

- q : charge [C] (**Coulomb**)
- i : current [A]
- e : voltage [V]
- R : resistance [Ω]
- C : capacitance [**Farad**]
- L : inductance [**H**] (**Henry**)
- p : power [**Watt**]
- w : work (energy) [J]
 $1 \text{ [J]} \text{ (Joule)} = 1 \text{ [V-A-sec]}$

$$\frac{d}{dt} q = i$$

$$q(t_1) = q(t_0) + \sum_{t_0}^{t_1} i(t) dt$$

$$p = e \cdot i$$

$$w(t_1) = w(t_0) + \sum_{t_0}^{t_1} p(t) dt$$

$$= w(t_0) + \sum_{t_0}^{t_1} (e \cdot i) dt$$

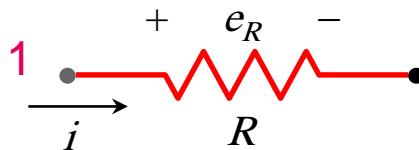
Basic Modeling Elements

- **Resistor**

- Ohms Law

Voltage across is proportional to the through current.

$$e_{12} = e_1 - e_2 = e_R = R i \Leftrightarrow i = \frac{1}{R} e_R$$



Static relation

- Dissipates energy through heat.

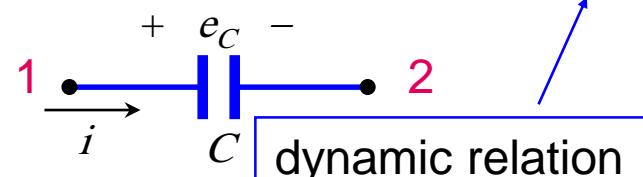
$$p = R i^2 = \frac{1}{R} e^2$$

- Analogous to friction elements in mechanical systems, e.g. dampers

- **Capacitor**

- Charge collected is proportional to the voltage across.
- Current is proportional to the rate of change of the voltage across.

$$q = C e_c \Leftrightarrow i = C \left(\frac{d}{dt} e_c \right) = C \left(\frac{d}{dt} e_{12} \right)$$



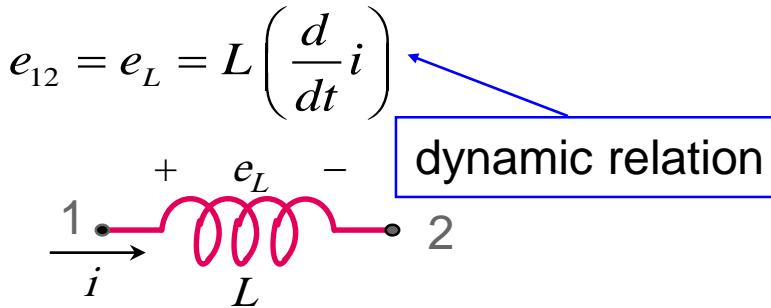
dynamic relation

- Energy supplied is stored in its electric field and can affect future circuit response.
- Steady-state response: $i=0$, Open Circuit

Basic Modeling Elements

- **Inductor**

- Voltage across is proportional to the rate of the change of the through current.



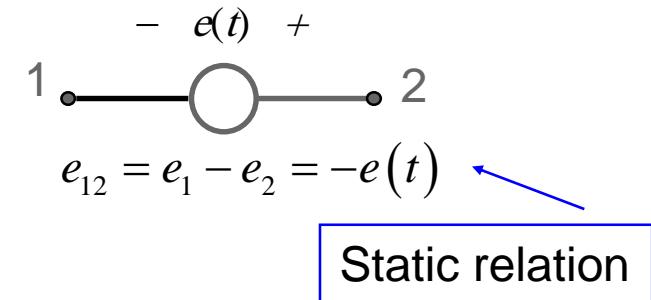
- Energy supplied is stored in its magnetic field.

$$w = \frac{1}{2} L i^2$$

- Steady-state response: $e=0$, Short Circuit

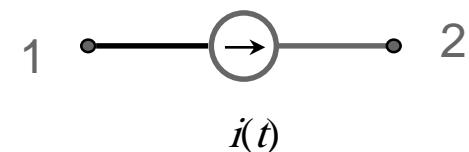
- **Voltage Source**

- Maintain specified voltage across two points, regardless of the required current.

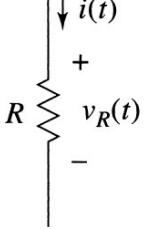
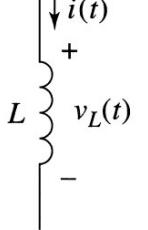
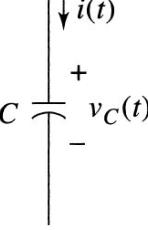
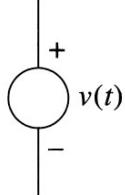
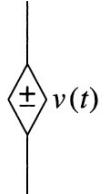
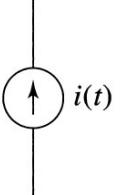
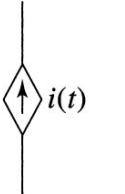


- **Current Source**

- Maintain specified current, regardless of the required voltage.



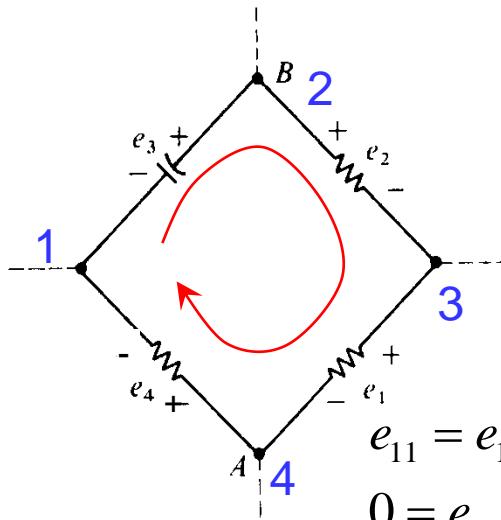
Basic Modeling Elements

Resistor	Inductor	Capacitor
		
$v_R(t) = Ri(t)$		$v_L(t) = L \frac{di}{dt}$
$i(t) = \frac{1}{R}v_R(t)$		$i(t) = \frac{1}{L} \int_{-\infty}^t v_L(t) dt$
Voltage Source		Current Source
	$v(t)$ a given function of time	
	$v(t)$ expressed in terms of other network voltages or currents	
		
		$i(t)$ a given function of time
		
		$i(t)$ expressed in terms of other network voltages or currents

Interconnection Laws

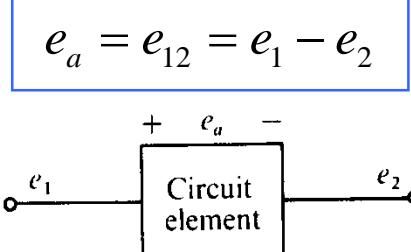
- **Kirchhoff's Voltage Law (loop law)**
 - The total voltage drop along any *closed loop* in the circuit is zero.
- **Kirchhoff's Current Law (node law)**
 - The algebraic sum of the currents at any node in the circuit is zero.

$$\sum_{\text{Closed Loop}} e_j = 0$$



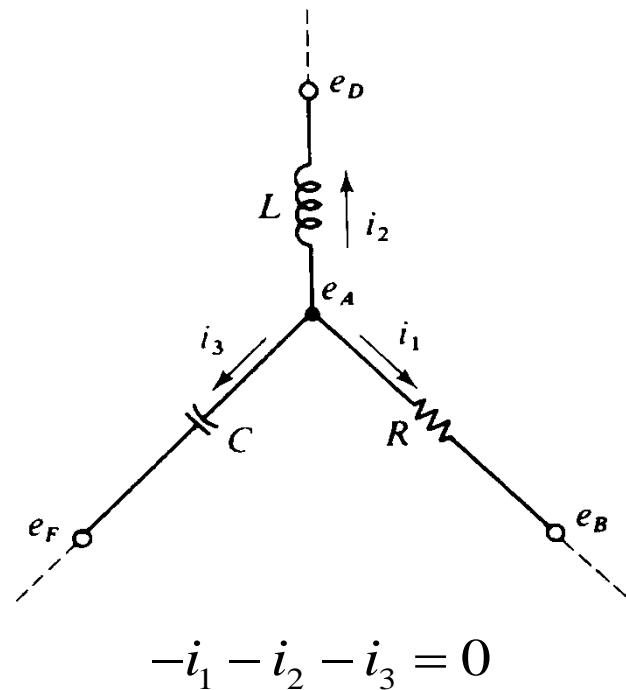
$$\begin{aligned} e_{11} &= e_1 - e_1 = 0 \\ 0 &= e_{11} = e_{12} + e_{23} + e_{34} + e_{41} \\ -e_3 + e_2 + e_1 + e_4 &= 0 \end{aligned}$$

$$e_{12} \quad e_{23} \quad e_{34} \quad e_{41}$$



$$e_a = e_{12} = e_1 - e_2$$

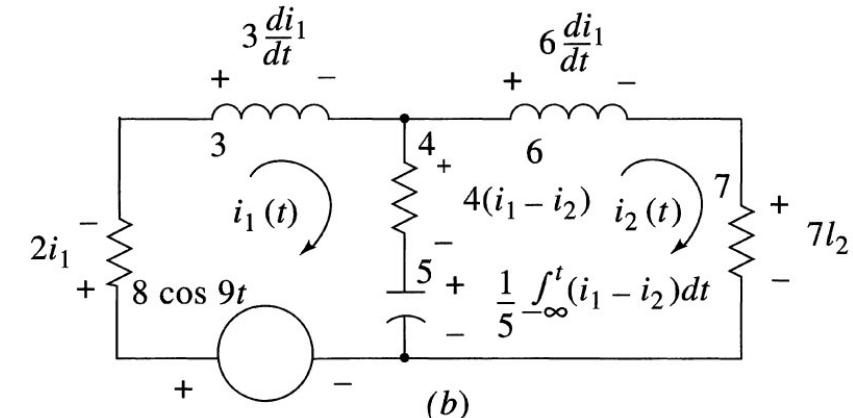
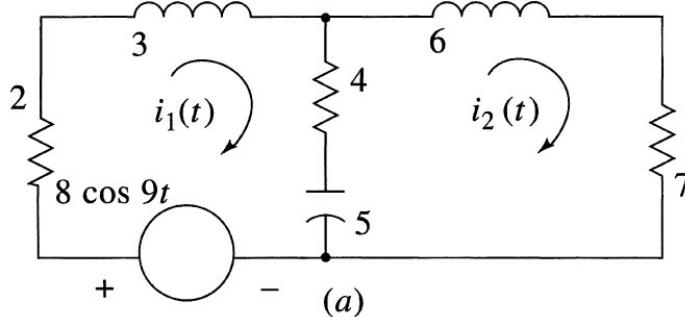
$$\sum_{\text{Any Node}} i_j = 0$$



$$-i_1 - i_2 - i_3 = 0$$

Basic Modeling Elements

Example 4



$$\text{loop1} \quad 2i_1(t) + 3\frac{di_1(t)}{dt} + 4(i_1(t) - i_2(t)) + \frac{1}{5} \int_0^t (i_1(t) - i_2(t)) dt = 8 \cos 9t$$

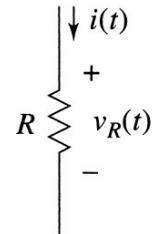
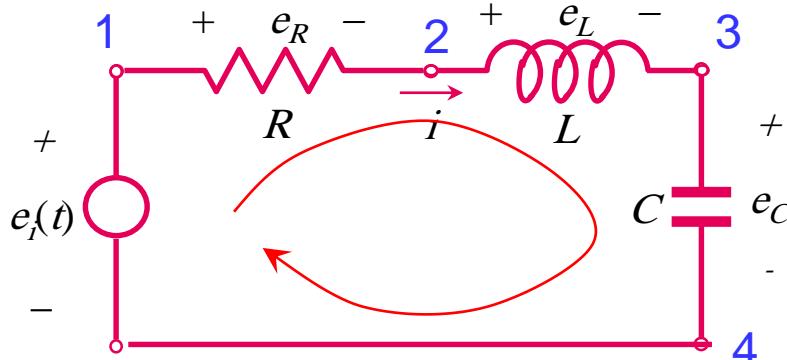
$$\text{loop2} \quad \frac{1}{5} \int_0^t (i_2(t) - i_1(t)) dt + 4(i_2(t) - i_1(t)) + 6\frac{di_2(t)}{dt} + 7i_2(t) = 0$$

Modeling Steps

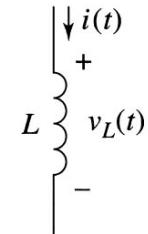
- ***Understand System Function and Identify Input/Output Variables***
- ***Draw Simplified Schematics Using Basic Elements***
- ***Develop Mathematical Model***
 - Label Each Element and the Corresponding Voltages.
 - Label Each Node and the Corresponding Currents.
 - Apply Interconnection Laws.
 - Check that the Number of Unknown Variables equals the Number of Equations
 - Eliminate Intermediate Variables to Obtain Standard Forms.

In Class Exercise

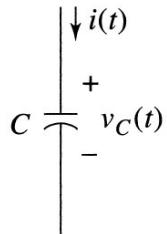
Derive the I/O model for the following circuit. Let voltage $e_i(t)$ be the input and the voltage across the capacitor be the output.



$$i(t) = \frac{1}{R}v_R(t)$$



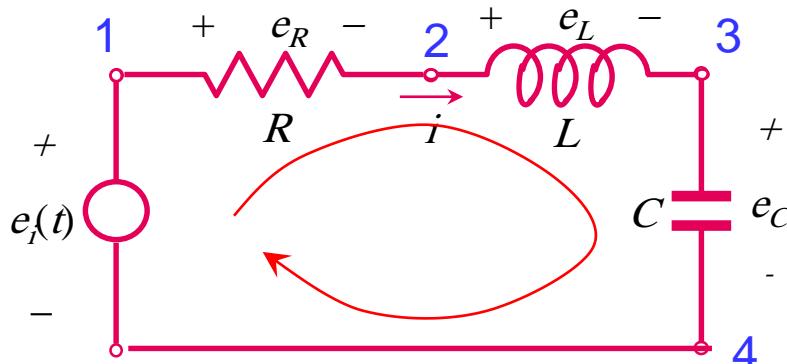
$$i(t) = \frac{1}{L} \int_{-\infty}^t v_L(t) dt$$



$$i(t) = C \frac{dv_c}{dt}$$

In Class Exercise

Derive the I/O model for the following circuit. Let voltage $e_i(t)$ be the input and the voltage across the capacitor be the output.



Element Laws:

$$\begin{aligned} e_{12} &= e_R = iR & e_{23} &= e_L = L \left(\frac{d}{dt} i \right) \\ i &= C \left(\frac{d}{dt} e_{34} \right) & e_{41} &= -e_i(t) \end{aligned}$$

Kirchhoff's Loop Law:

$$e_{12} + e_{23} + e_{34} + e_{41} = 0$$

No. of Unknowns:

$$e_{12}, \quad e_{23}, \quad y = e_{34}, \quad e_{41}, \quad i$$

Simplify

$$\begin{cases} i = C \frac{de_{34}}{dt} \\ iR + L \frac{di}{dt} + e_{34} - e_i = 0 \end{cases}$$

I/O Model:

$$LC \frac{d^2 e_{34}}{dt^2} + RC \frac{de_{34}}{dt} + e_{34} = e_i$$

How to get I/O model by concept of complex impedance ?

Mechanical translational system

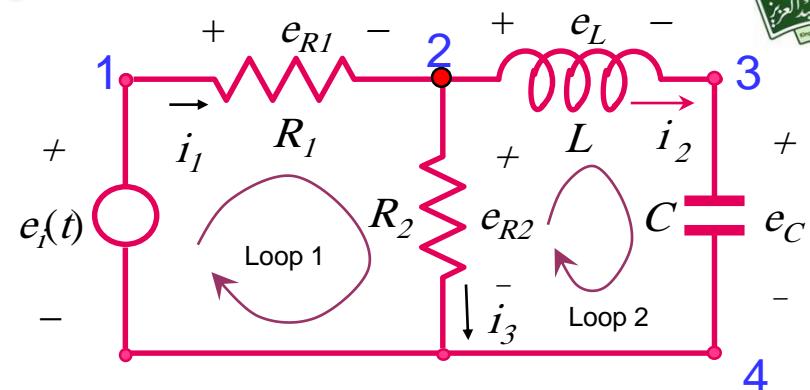
$$M \frac{d^2 x}{dt^2} + B \frac{dx}{dt} + Kx = f$$

Mechanical rotational system

$$I_c \frac{d^2 \theta}{dt^2} + B \frac{d\theta}{dt} + K\theta = \tau$$

POP. Quiz

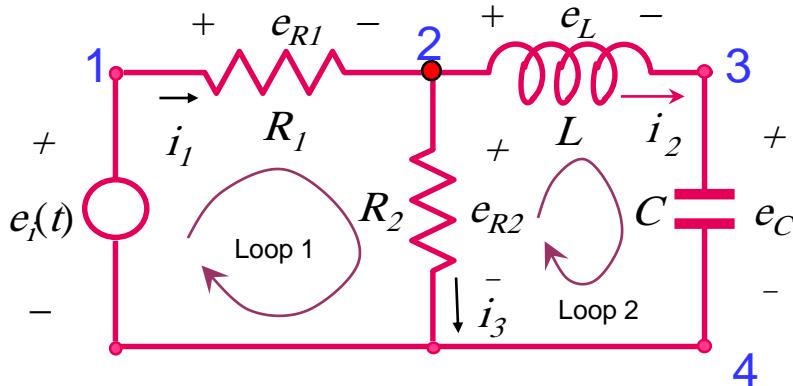
Obtain the I/O model for the following circuit. The input is the voltage $e_i(t)$ of the voltage source and the through current of the inductor is the output.



Component	Voltage-current	Current-voltage	Voltage-charge	Impedance $Z(s) = V(s)/I(s)$	Admittance $Y(s) = I(s)/V(s)$
 Capacitor	$v(t) = \frac{1}{C} \int_0^t i(\tau) d\tau$	$i(t) = C \frac{dv(t)}{dt}$	$v(t) = \frac{1}{C} q(t)$	$\frac{1}{Cs}$	Cs
 Resistor	$v(t) = Ri(t)$	$i(t) = \frac{1}{R} v(t)$	$v(t) = R \frac{dq(t)}{dt}$	R	$\frac{1}{R} = G$
 Inductor	$v(t) = L \frac{di(t)}{dt}$	$i(t) = \frac{1}{L} \int_0^t v(\tau) d\tau$	$v(t) = L \frac{d^2q(t)}{dt^2}$	Ls	$\frac{1}{Ls}$

Example

Obtain the I/O model for the following circuit. The input is the voltage $e_i(t)$ of the voltage source and the through current of the inductor is the output.



Elemental Equations:

$$e_{12} = e_{R1} = i_1 R_1, \quad e_{23} = e_L = L \left(\frac{d}{dt} i_2 \right)$$

$$i_2 = C \left(\frac{d}{dt} e_{34} \right), \quad e_{24} = -e_{42} = e_{R2} = i_3 R_2$$

$$e_{41} = -e_i(t)$$

Voltage Law

$$\text{Loop 1: } e_{12} + e_{24} + e_{41} = 0$$

$$\text{Loop 2: } e_{23} + e_{34} + e_{42} = 0$$

Current Law

$$\text{Node 2: } i_1 - i_2 - i_3 = 0$$

Unknown Variables

$$e_{12}, \quad e_{24}, \quad e_{41}, \quad e_{23}, \quad e_{34}, \quad e_{42},$$

$$i_1, \quad y = i_2, \quad i_3.$$

I/O Model

$$LC \frac{d^2 i_2}{dt^2} + \frac{R_1 R_2 C}{R_1 + R_2} \frac{di_2}{dt} + i_2 = \frac{R_2 C}{R_1 + R_2} \frac{de_i}{dt}$$

Example (cont.)

$$\begin{cases} i_2 = C \frac{de_{34}}{dt} \\ i_1 R_1 + i_3 R_2 - e_i = 0 \\ L \frac{di_2}{dt} + e_{34} - i_3 R_2 = 0 \\ i_1 = i_2 + i_3 \end{cases} \Rightarrow$$

$$\begin{cases} i_2 = C \frac{de_{34}}{dt} \\ i_2 R_1 + i_3 (R_1 + R_2) - e_i = 0 \\ LC \frac{d^2 i_2}{dt^2} + C \frac{de_{34}}{dt} - \frac{di_3}{dt} CR_2 = 0 \end{cases} \Rightarrow$$

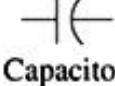
$$\begin{cases} i_3 = \frac{-1}{R_1 + R_2} (i_2 R_1 - e_i) \\ LC \frac{d^2 i_2}{dt^2} + i_2 + CR_2 \frac{d}{dt} \underbrace{\frac{1}{R_1 + R_2}}_{i_3} (i_2 R_1 - e_i) = 0 \end{cases} \Rightarrow LC \frac{d^2 i_2}{dt^2} + i_2 + CR_2 \frac{d}{dt} \underbrace{\frac{1}{R_1 + R_2}}_{i_3} (i_2 R_1 - e_i) = 0$$

$$LC \frac{d^2 i_2}{dt^2} + \frac{R_1 R_2 C}{R_1 + R_2} \frac{di_2}{dt} + i_2 = \frac{R_2 C}{R_1 + R_2} \frac{de_i}{dt}$$

Transfer Fns of Electrical Nets

Basic relationship for electrical components

TABLE 2.3 Voltage-current, voltage-charge, and impedance relationships for capacitors, resistors, and inductors

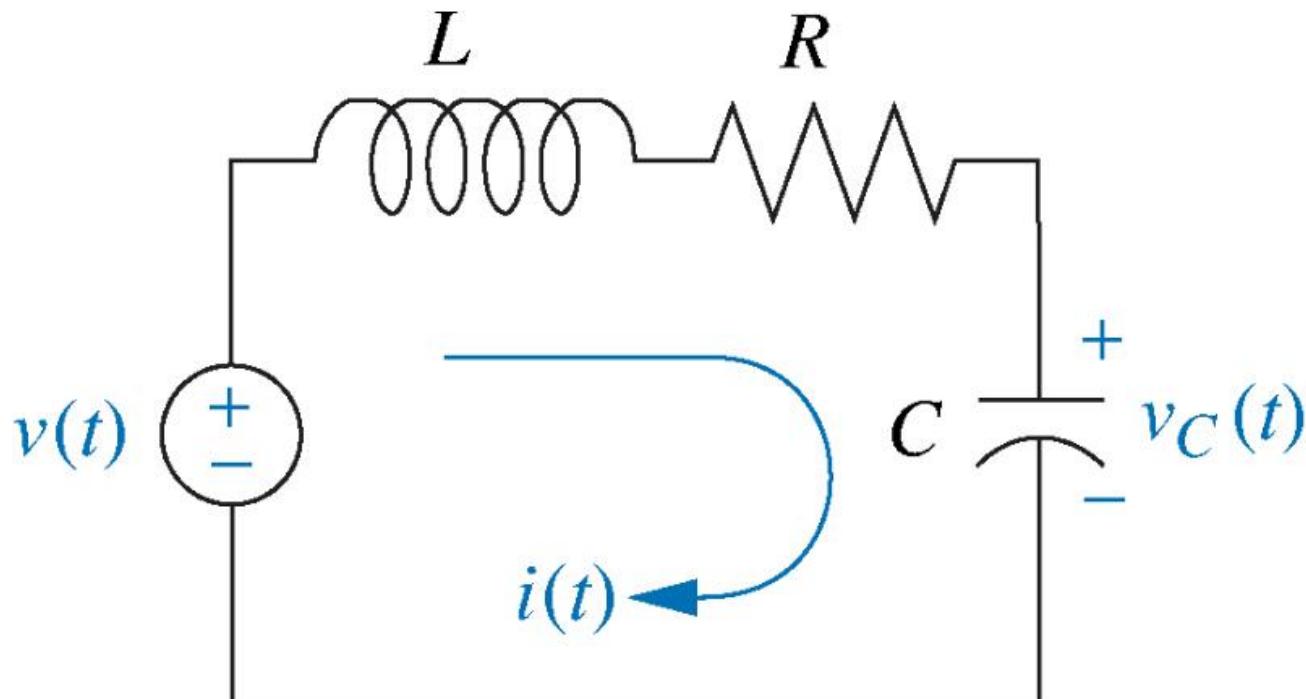
Component	Voltage-current	Current-voltage	Voltage-charge	Impedance $Z(s) = V(s)/I(s)$	Admittance $Y(s) = I(s)/V(s)$
 Capacitor	$v(t) = \frac{1}{C} \int_0^t i(\tau) d\tau$	$i(t) = C \frac{dv(t)}{dt}$	$v(t) = \frac{1}{C} q(t)$	$\frac{1}{Cs}$	Cs
 Resistor	$v(t) = Ri(t)$	$i(t) = \frac{1}{R} v(t)$	$v(t) = R \frac{dq(t)}{dt}$	R	$\frac{1}{R} = G$
 Inductor	$v(t) = L \frac{di(t)}{dt}$	$i(t) = \frac{1}{L} \int_0^t v(\tau) d\tau$	$v(t) = L \frac{d^2q(t)}{dt^2}$	Ls	$\frac{1}{Ls}$

$v(t) = V$ (volts), $i(t) = \text{current} = A$ (Amp), $q(t) = Q$ (coulombs), $C = F$ (farads), $R = \Omega$ (ohms), $L = H$ (henries)

Circuit Analysis – Mesh Analysis

Example 2.6 (Diff. Eqn.)

Find the transfer function V_C/V .

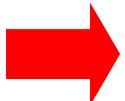


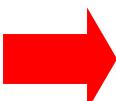
Circuit Analysis – Mesh Analysis

Solution

Summing the voltage around the loop gives:

$$L \frac{di}{dt} + Ri + \frac{1}{C} \int idt = v$$

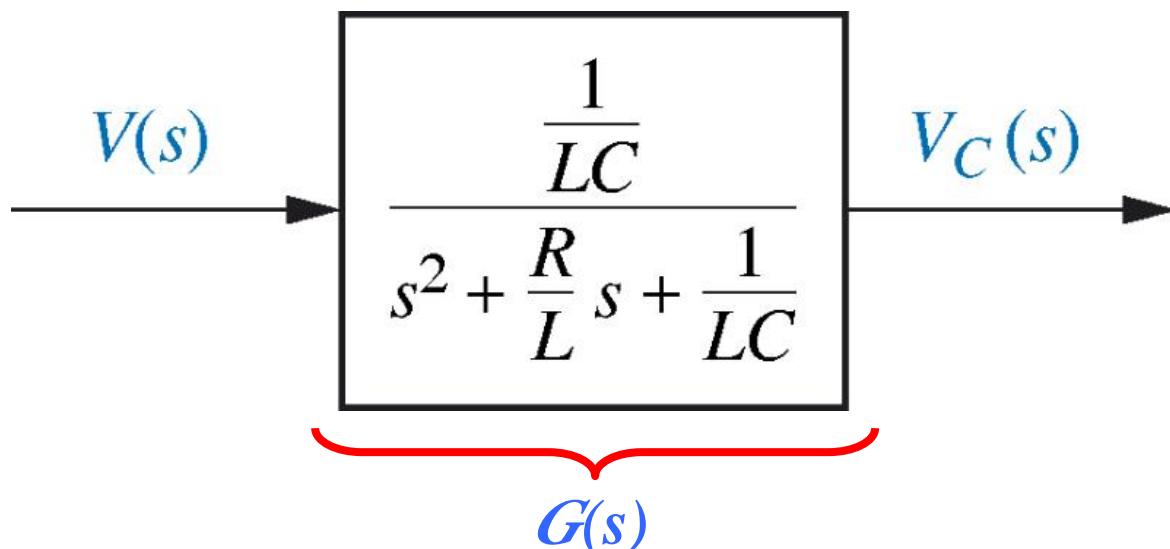
But, $i = \frac{dq}{dt}$  $L \frac{d^2q}{dt^2} + R \frac{dq}{dt} + \frac{1}{C} q = v$

as $q = Cv_c$  $LC \frac{d^2v_c}{dt^2} + RC \frac{dv_c}{dt} + v_c = v$

Circuit Analysis – Mesh Analysis

Applying the Laplace Transform, gives

$$(LCs^2 + RCs + 1)V_c(s) = V(s)$$

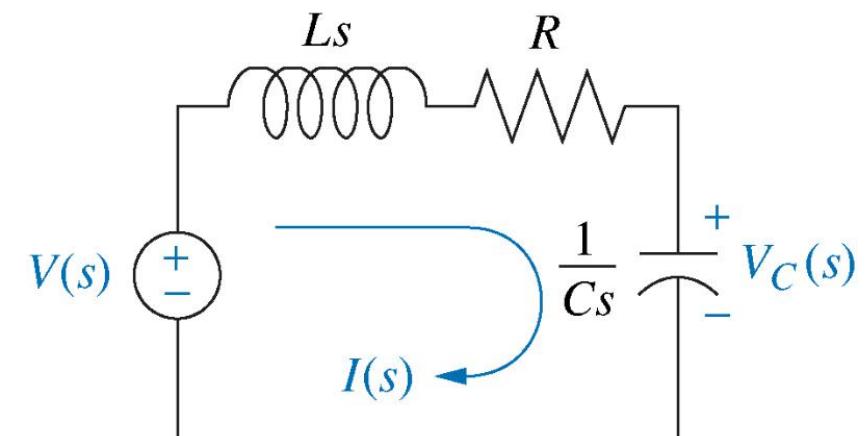
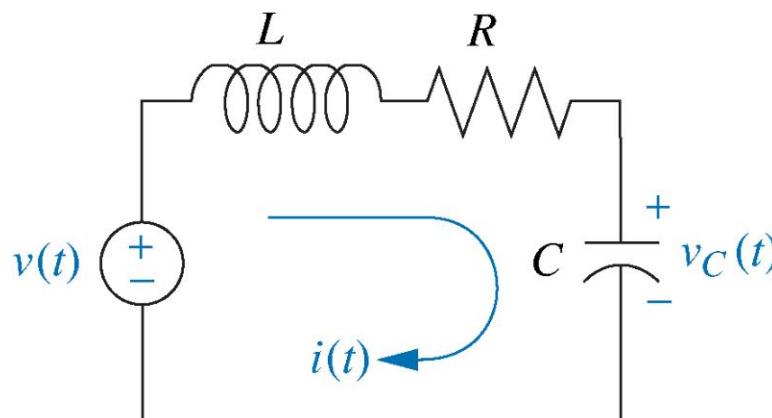


Component Transfer Function

Capacitance (C) $\rightarrow V(s) = \frac{1}{Cs} I(s)$

Resistance (R) $\rightarrow V(s) = RI(s)$ $\rightarrow \frac{V(s)}{I(s)} = Z(s)$

Inductance (L) $\rightarrow V(s) = LsI(s)$



Complex impedance $Z(s)$

Ratio of $E(s)$, the Laplace transform (LP) of voltage across the terminal, to $I(s)$, the Laplace transform (LP) of current through the element, under the assumption of zero initial conditions

$$Z(s) = \left. \frac{E(s)}{I(s)} \right|_{I.C.s=0}$$

R , $\frac{1}{Cs}$, and Ls , respectively.

Complex impedances of resistor, capacitor, inductor are

Series and Parallel Elements

- Parallel combinations**

Same Voltage across elements
 $\Delta(Voltage_i) = \Delta(Voltage_j)$

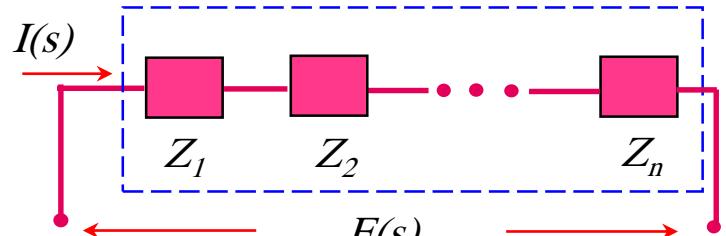
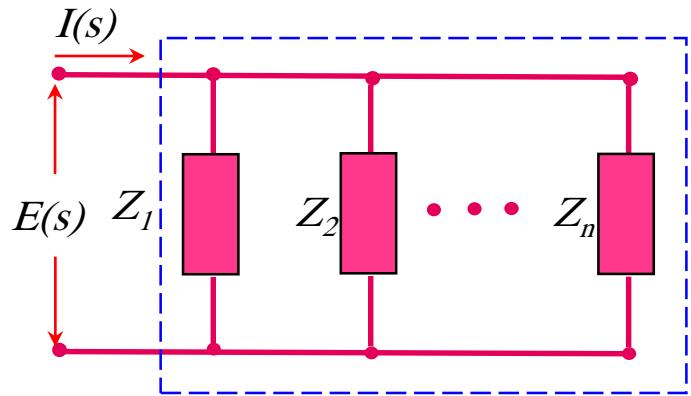
- Series combinations**

Same Current through elements
 $(Current_i) = (Current_j)$

- Equivalent Complex Impedance**

$$\frac{1}{Z_{parallel}(s)} = \frac{1}{Z_1(s)} + \frac{1}{Z_2(s)} + \dots + \frac{1}{Z_n(s)}$$

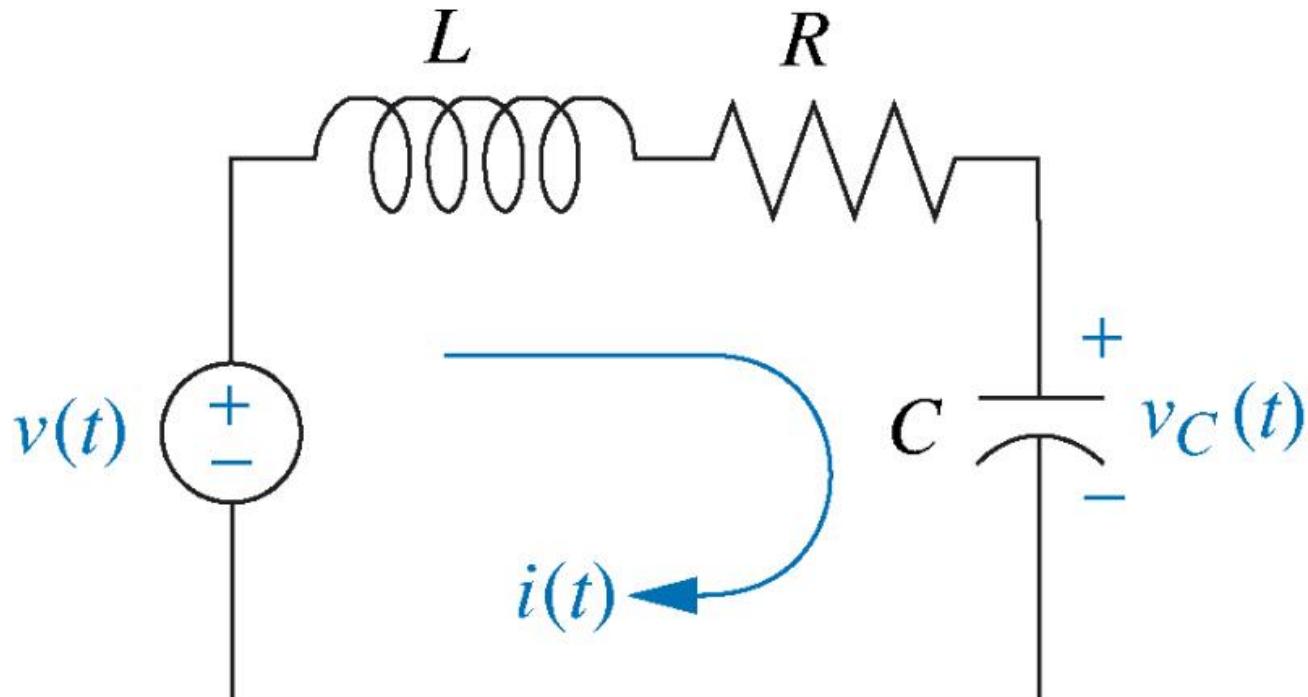
$$Z_{series}(s) = Z_1(s) + Z_2(s) + \dots + Z_n(s)$$



Circuit Analysis – Mesh Analysis

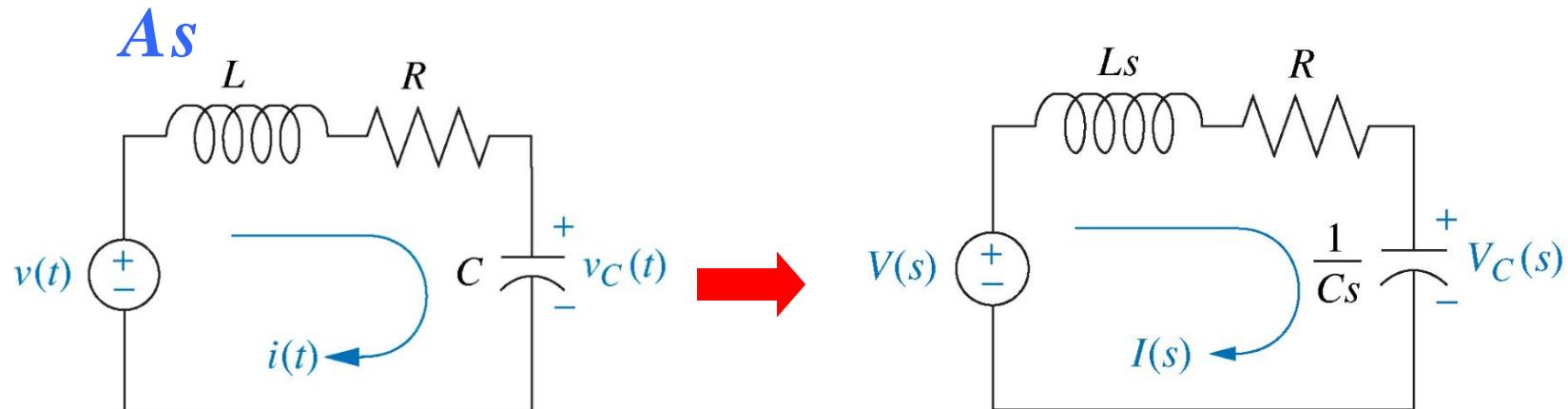
Example (Transform Methods):

Find the transfer function V_C/V by mesh analysis & transform methods.



Circuit Analysis – Mesh Analysis

Solution



Then, as $Z(s) = Ls + R + \frac{1}{Cs}$, $\frac{V(s)}{I(s)} = Z(s)$ & $\frac{V_c(s)}{I(s)} = \frac{1}{Cs}$

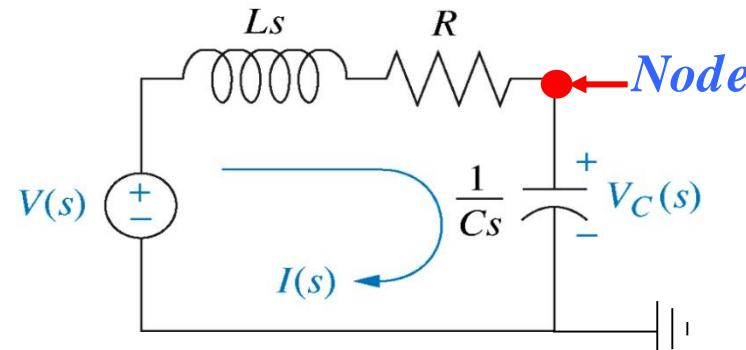


$$\frac{V_c(s)}{V(s)} = \frac{1}{(Ls^2 + Rs + 1)}$$

Circuit Analysis – Nodal Analysis

Example 2.8

Find the transfer function V_c/V by nodal analysis & transform methods.



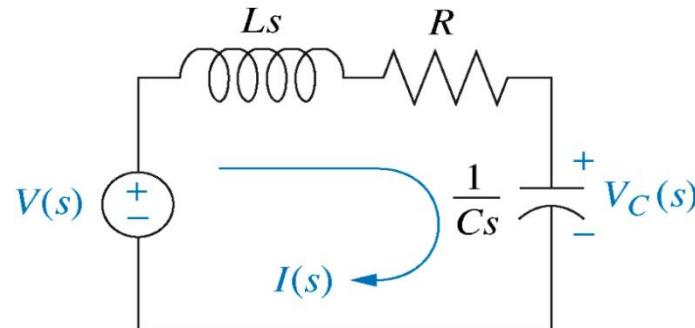
Then, as $I(s) = \frac{V(s)}{Z(s)}$ → *Kirchoff's Current Law at node*

$$\rightarrow \frac{V_c(s)}{\frac{1}{Cs}} + \frac{V_c(s) - V(s)}{Ls + R} = 0 \quad \&$$

$$\frac{V_c(s)}{V(s)} = \frac{1}{(LCs^2 + RCs + 1)}$$

Example 2.9

Find the transfer function V_c/V by voltage division & transform methods.



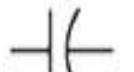
Voltage across capacitor is a portion of the input voltage which is proportional to the capacitor impedance to the sum of impedances.



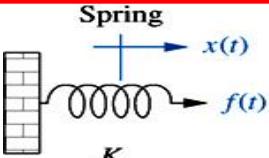
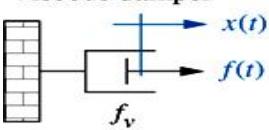
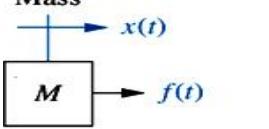
$$V_c(s) = \frac{1/C_s}{(Ls + R + 1/C_s)} V(s)$$

Basic Relationship for Electrical & Mechanical Components

Electrical Components

Component	Voltage-current	Current-voltage
 Capacitor	$v(t) = \frac{1}{C} \int_0^t i(\tau) d\tau$	$i(t) = C \frac{dv(t)}{dt}$
 Resistor	$v(t) = R i(t)$	$i(t) = \frac{1}{R} v(t)$
 Inductor	$v(t) = L \frac{di(t)}{dt}$	$i(t) = \frac{1}{L} \int_0^t v(\tau) d\tau$

Mechanical Components

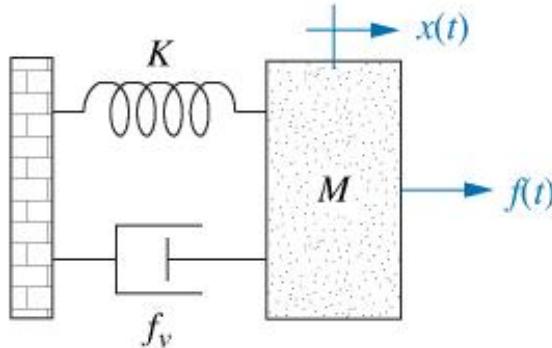
Component	Force-velocity
 Spring	$f(t) = K \int_0^t v(\tau) d\tau$
 Viscous damper	$f(t) = f_v v(t)$
 Mass	$f(t) = M \frac{dv(t)}{dt}$

Two analogs

- **Voltage-current** → **Force-Velocity**
- **Current-voltage** → **Force-Velocity**

Electric Circuit Analogs

Series Analog

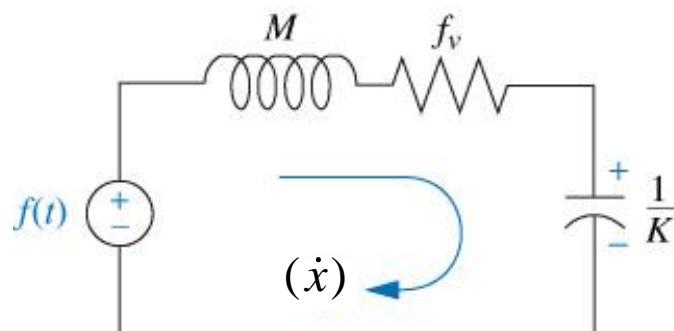


$$M\ddot{x} + f_v \dot{x} + Kx = f$$

Voltage (v)-current (i) → Force (f)-Velocity (\dot{x})



$$M \frac{di}{dt} + f_v i + K \int i dt = v$$



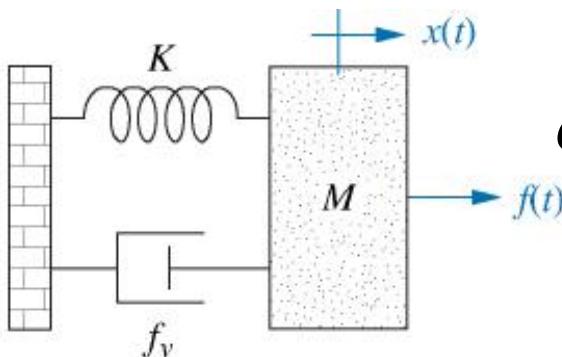
mass = M → inductor = M henries

viscous damper = f_v → resistor = f_v ohms

spring = K → capacitor = $\frac{1}{K}$ farads

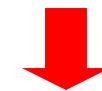
Electric Circuit Analogs

Parallel Analog

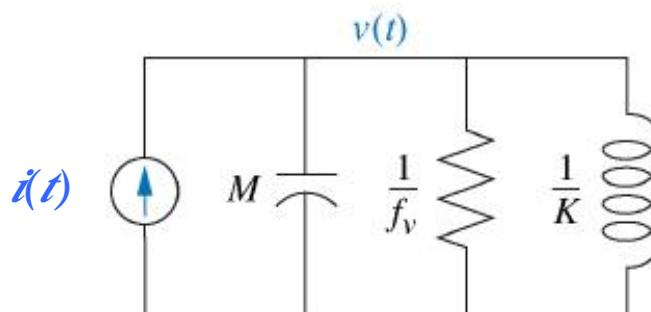
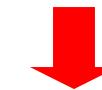


$$M\ddot{x} + f_v \dot{x} + Kx = f$$

Current (i) -voltage (v) → Force (f)-Velocity (x)



$$M \frac{dv}{dt} + f_v v + K \int v dt = i$$

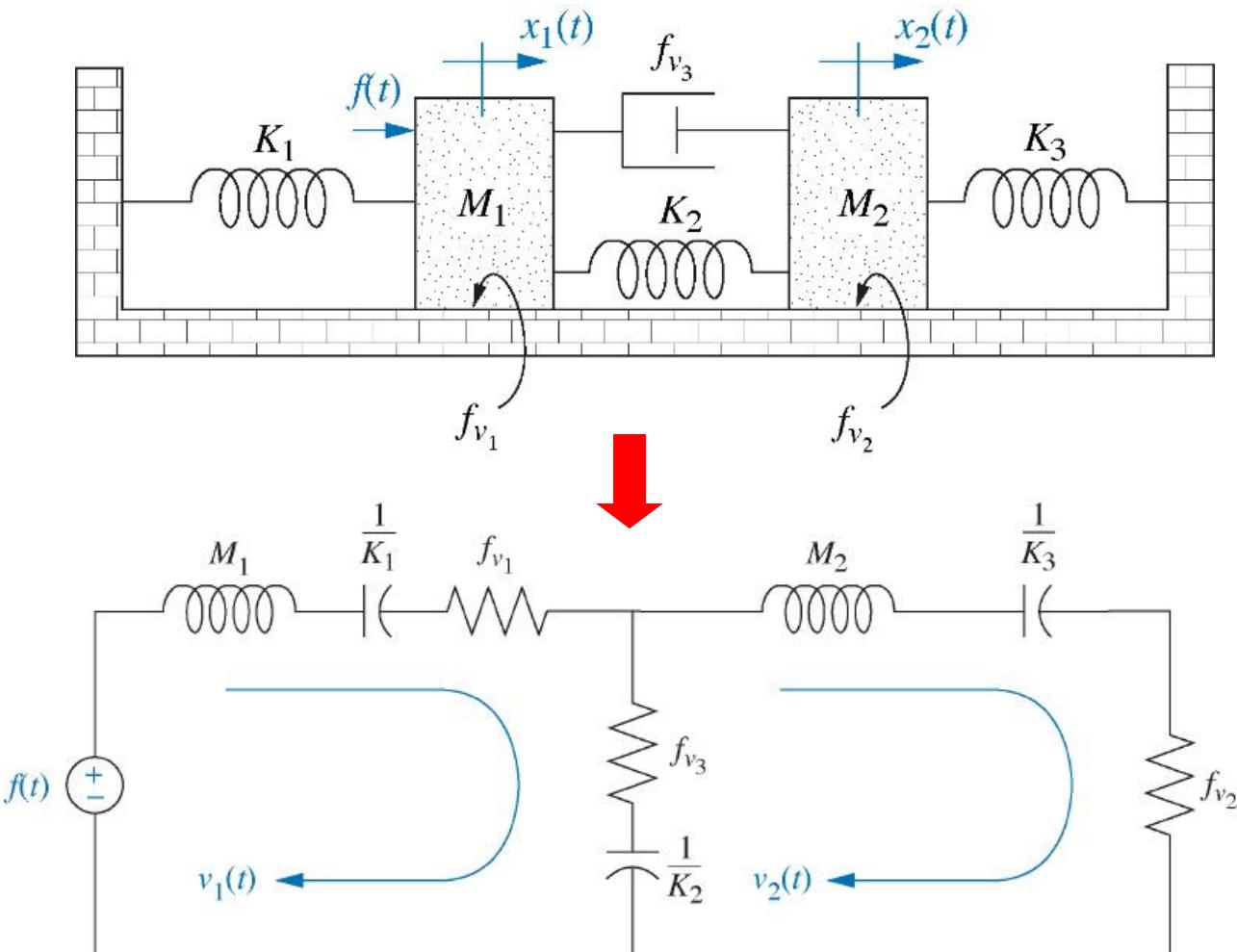


mass = $M \rightarrow$ capacitor = M farads

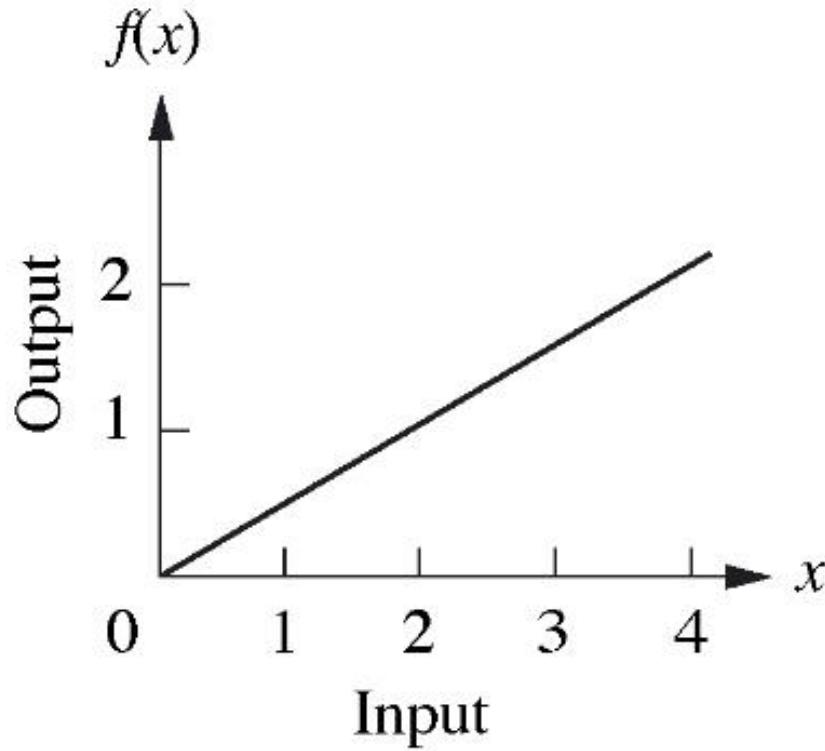
viscous damper = $f_v \rightarrow$ resistor = $\frac{1}{f_v}$ ohms

spring = $K \rightarrow$ inductor = $\frac{1}{K}$ henries

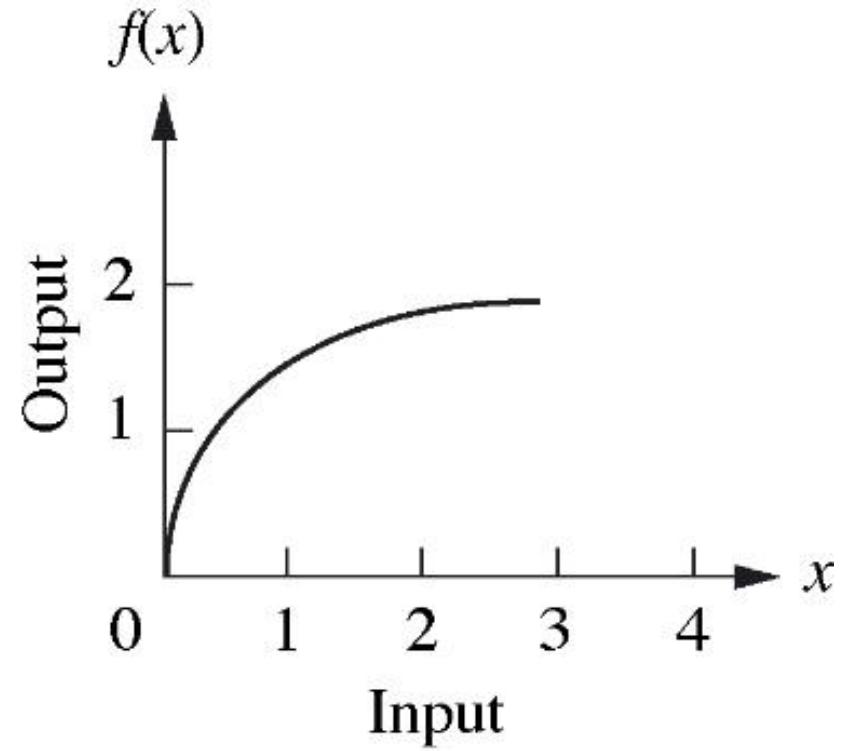
Electric Circuit Analogs



Nonlinearities

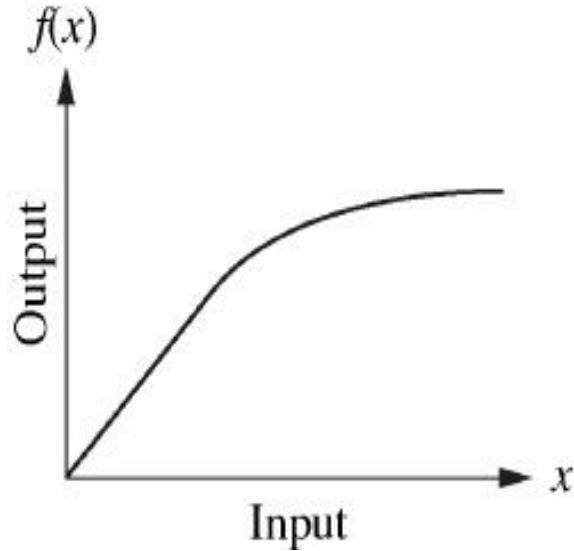


Linear System

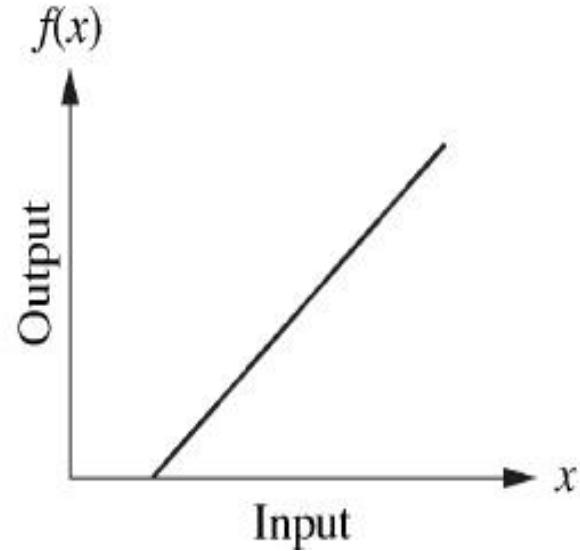


Nonlinear System

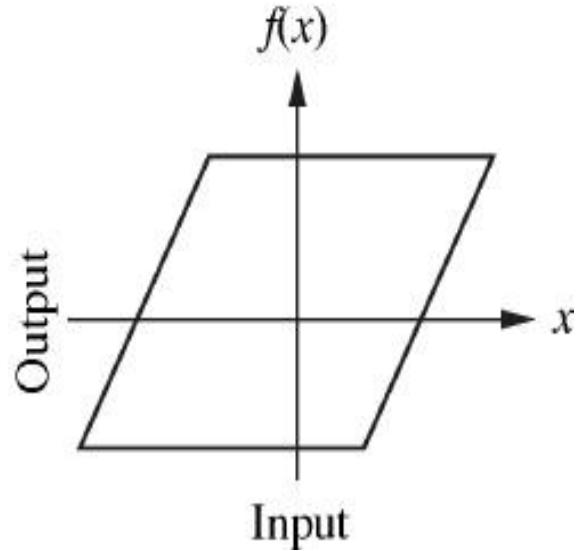
Nonlinearities



Amplifier Saturation



Motor Dead Zone



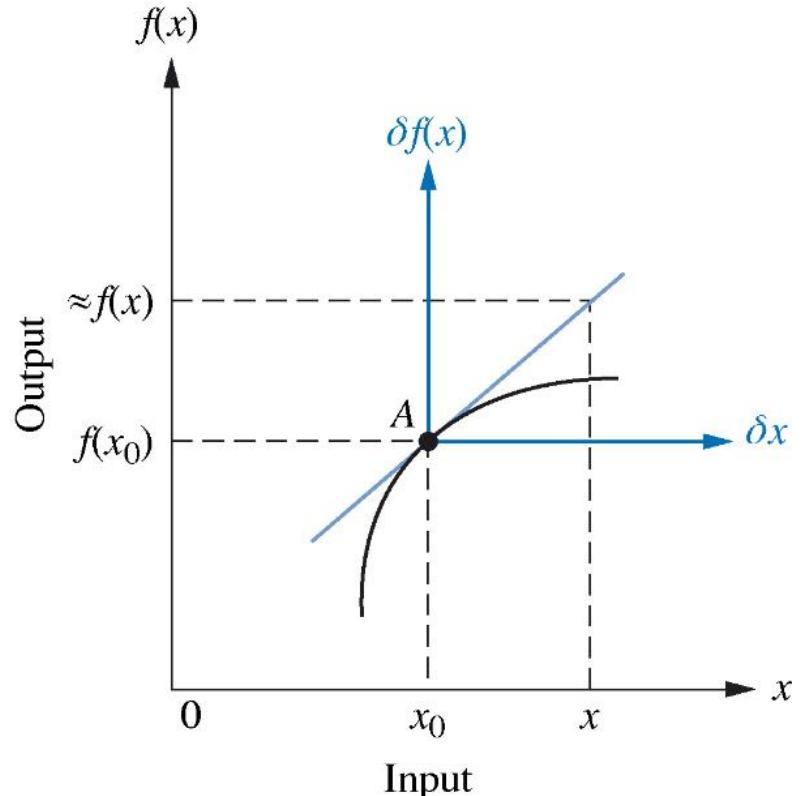
Backlash in Gears

Linearization of Nonlinearities

Using Taylor Series Expansion

$$f(x) = f(x_0) + \frac{df}{dx} \Big|_{x=x_0} \frac{(x - x_0)}{1!} + \frac{d^2 f}{dx^2} \Big|_{x=x_0} \frac{(x - x_0)^2}{2!} + \dots$$

$$\approx f(x_0) + \frac{df}{dx} \Big|_{x=x_0} \frac{(x - x_0)}{1!}$$



→ $f(x) - f(x_0) = \frac{df}{dx} \Big|_{x=x_0} (x - x_0)$ →

$$\delta f(x) = \frac{df}{dx} \Big|_{x=x_0} \delta x$$

Linearization of Nonlinearities

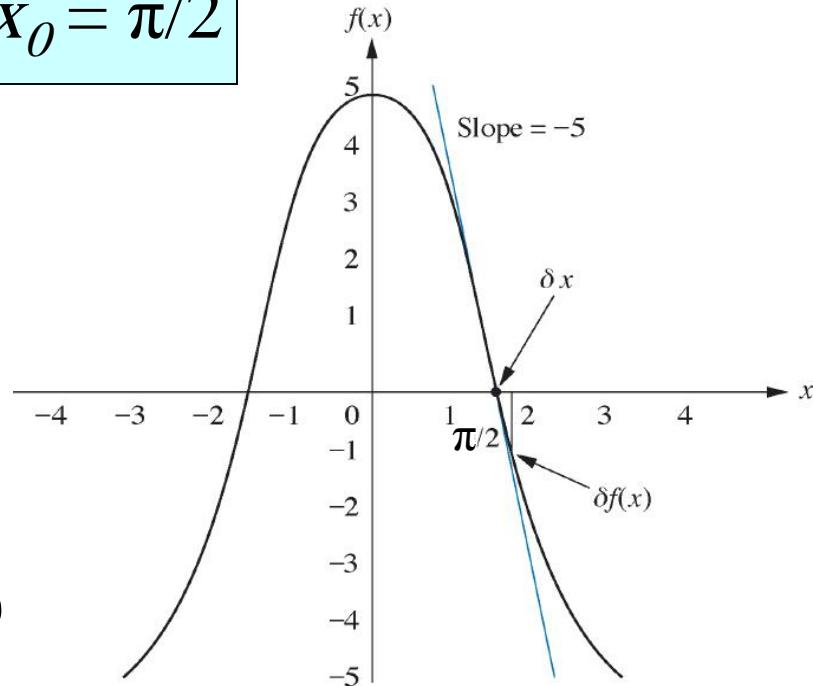
Linearize $f(x) = 5 \cos(x)$ about $x_0 = \pi/2$

As

$$f(x) - f(x_0) \cong \frac{df}{dx} \Big|_{x=x_0} (x - x_0)$$

→ $f(x) = 5 \cos(x) = f(x_0) + \frac{df}{dx} \Big|_{x=x_0} (x - x_0)$

$$\begin{aligned} &= 5 \cos(\pi/2) + (-5 \sin x)_{x=\pi/2} (x - \pi/2) \\ &= -5(x - \pi/2) \end{aligned}$$



Linearization of Nonlinearities

Linearize the following system about $x_0 = \pi/4$

$$\ddot{x} + 2\dot{x} + \cos x = 0$$

Let $x = \delta x + \pi/4$

→ $\delta\ddot{x} + 2\delta\dot{x} + \cos(\delta x + \pi/4)$
 $= \delta\ddot{x} + 2\delta\dot{x} + \cos(\pi/4) - \sin(\pi/4)\delta x = 0$

or $\delta\ddot{x} + 2\delta\dot{x} - 0.707\delta x = -0.707$

→ $\ddot{y} + 2\dot{y} - 0.707 y = -0.707$ with $y = \delta x$

Linearization of Nonlinearities

Find the transfer function V_L/V

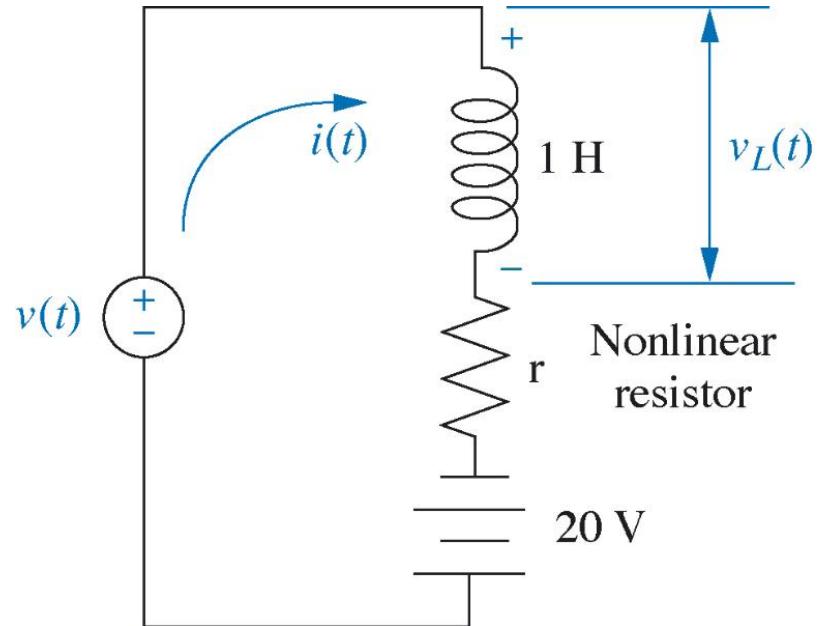
Solution

$$L \frac{di}{dt} + 10 \ln\left(\frac{1}{2}i\right) - 20 = v$$

→ *Equilibrium attained when*

$$10 \ln\left(\frac{1}{2}i_0\right) - 20 = 0$$

→ $i_0 = 14.78 \text{ Amp.}$



Nonlinear Resistor

$$i = 2e^{0.1v} \rightarrow v = 10 \ln\left(\frac{1}{2}i\right)$$

Linearization of Nonlinearities

Linearize around equilibrium $i = \delta i + 14.78$

$$\begin{aligned} & \rightarrow L \frac{d(i_0 + \delta i)}{dt} + 10 \ln \frac{1}{2}(i_0 + \delta i) - 20 \\ &= L \frac{d\delta i}{dt} + 10 \left(\ln \frac{1}{2} i_0 + \frac{1}{i_0} \delta i \right) - 20 \\ &= L \frac{d\delta i}{dt} + 0.677 \delta i = v \end{aligned}$$

Applying Laplace Transform

$$\rightarrow \delta i(s) = \frac{V(s)}{s + 0.677}$$

Linearization of Nonlinearities

But,

$$v_L = L \frac{di}{dt} = L \frac{d(i_0 + \delta i)}{dt} = L \frac{d\delta i}{dt}$$

Applying Laplace Transform



$$\delta i(s) = V_L / Ls$$

Combining with,

$$\delta i(s) = \frac{V(s)}{s + 0.677}$$



$$\frac{V_L}{V} = \frac{Ls}{s + 0.677}$$

END