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King Abdulaziz University
Engineering College
Department of Production and Mechanical System Design



MENG 366 Automatic Control

Final Exam
Closed-book Exam
Saturday: 25/11/1424 H
Time Allowed: Two Hours

Name:	Sec. No.:	ID No.:
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Question 1		10
Question 2		10
Question 3		10
Question 4		10
Question 5		10
TOTAL		50

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Instructions

1. There are totally 5 problems in this exam.
2. Show all work for partial credit.
3. Assemble your work for each problem in logical order.
4. Justify your conclusion. I cannot read minds.

1. Are these statements **true** or **false**?

	True	False
a. The initial value theorem relates the steady state behavior of $sF(s)$ in the neighborhood of $s=0$.	<input type="checkbox"/>	<input checked="" type="checkbox"/>
b. In control theory, the transfer functions are commonly used to characterize the input-output relationship of linear time-invariant systems	<input checked="" type="checkbox"/>	<input type="checkbox"/>
c. The transient response of a practical control system often exhibits damped oscillations before reaching a steady state.	<input checked="" type="checkbox"/>	<input type="checkbox"/>
d. A linear time-invariant control system is stable if the output eventually comes back to its equilibrium state when the system is subjected to initial conditions.	<input checked="" type="checkbox"/>	<input type="checkbox"/>
e. The open-loop poles are the roots of the characteristics equation.	<input type="checkbox"/>	<input checked="" type="checkbox"/>
f. The main advantage of using the Bode plot is that multiplication of magnitudes can be converted into addition.	<input checked="" type="checkbox"/>	<input type="checkbox"/>
g. Nyquist plot are polar plots, while Bode plots are rectangular plots.	<input checked="" type="checkbox"/>	<input type="checkbox"/>
h. The Nyquist stability criterion determines the stability of an open-loop system from its closed-loop poles.	<input type="checkbox"/>	<input checked="" type="checkbox"/>
i. If there is one clockwise encirclement of Nyquist contour of $G(s)H(s)$ of the $-1+0j$ point, the system is NOT stable.	<input checked="" type="checkbox"/>	<input type="checkbox"/>
j. While conventional control theory is based on the transfer function, modern control theory is based on the description of system in state space.	<input checked="" type="checkbox"/>	<input type="checkbox"/>

2. Consider the system shown in Figure (1)

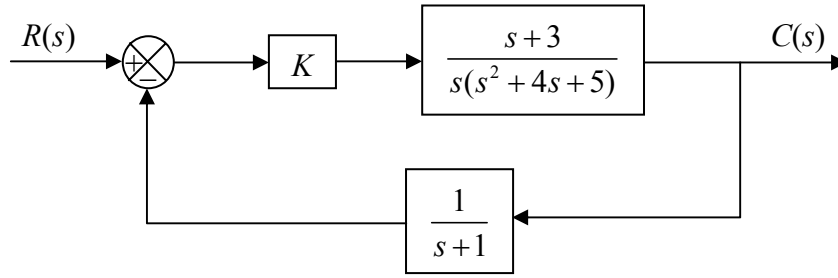


Figure (1)

a) Find the open-loop transfer function of the system.

The the open-loop transfer function of the system is $G(s)H(s) = \frac{K(s+3)}{s(s+1)(s^2+4s+5)}$

b) Find the closed-loop transfer function of the system.

The closed-loop transfer function of the system is:

$$\frac{G(s)}{1+G(s)H(s)} = \frac{\frac{K(s+3)}{s(s^2+4s+5)}}{1 + \frac{K(s+3)}{s(s+1)(s^2+4s+5)}} = \frac{K(s+3)(s+1)}{s(s+1)(s^2+4s+5) + K(s+3)}$$

After simplifications:

$$\frac{G(s)}{1+G(s)H(s)} = \frac{K(s+3)(s+1)}{s^4 + 5s^3 + 9s^2 + (5+K)s + 3K}$$

c) Find the characteristics equation of the system.

The characteristics equation is $s^4 + 5s^3 + 9s^2 + (5+K)s + 3K = 0$

d) Using *Routh's stability criterion*, determine all values of K for which the system is STABLE.

s^4	1	9	$3K$
s^3	5	$5+K$	
s^2	$\frac{40-K}{5}$	$3K$	
s	$(5+K) - \frac{75K}{40-K}$	0	
s^0	$3K$		

$$\frac{40-K}{5} > 0 \quad \text{or,} \quad K < 40$$

$$(5+K) - \frac{75K}{40-K} > 0 \quad \text{or,} \quad -44.49 < K < 4.49$$

$$3K > 0 \quad \text{or,} \quad K > 0$$

The stability condition is $0 < K < 4.49$

3. Obtain a state space representation of the mechanical system shown below in Figure (2) where u_1 and u_2 are inputs and y_1 and y_2 are outputs.

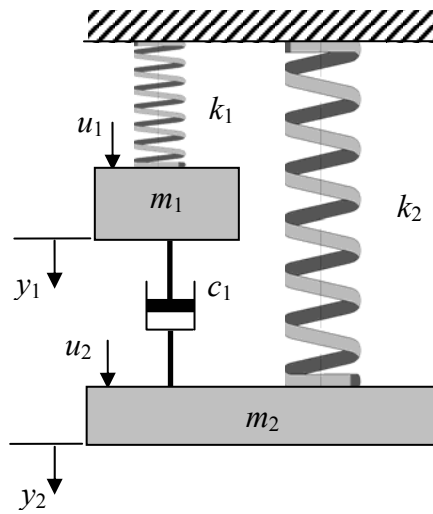
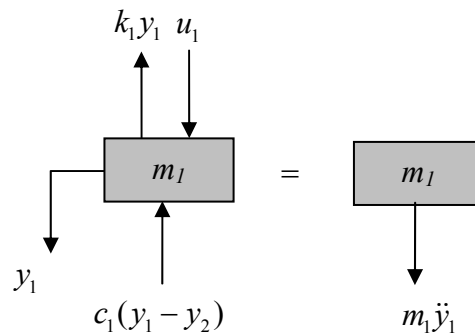


Figure (2)

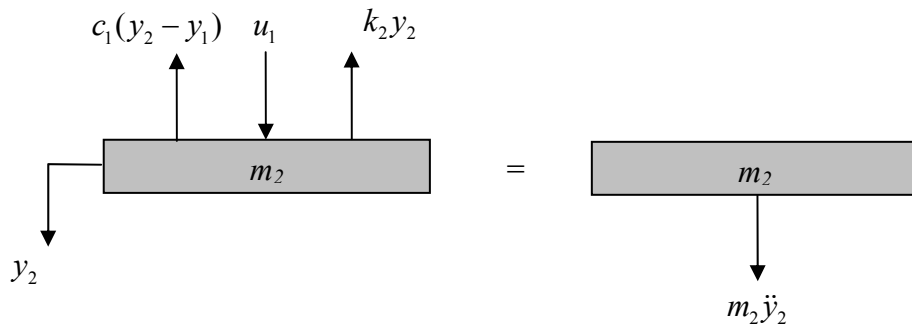
Solutions:

$$\sum \vec{F} = m\vec{a}$$



$$+k_1 y_1 + c_1 (\dot{y}_1 - \dot{y}_2) - u_1 = -m_1 \ddot{y}_1$$

$$m_1 \ddot{y}_1 + c_1 \dot{y}_1 + k_1 y_1 - c_1 \dot{y}_2 = u_1$$



$$k_2 y_2 - c_1 (\dot{y}_1 - \dot{y}_2) - u_2 = -m_2 \ddot{y}_2$$

$$m_2 \ddot{y}_2 + c_1 \dot{y}_2 + k_2 y_2 - c_1 \dot{y}_1 = u_2$$

Hence, the two equations of motion are:

$$m_1 \ddot{y}_1 + c_1 \dot{y}_1 + k_1 y_1 - c_1 \dot{y}_2 = u_1$$

$$m_2 \ddot{y}_2 + c_1 \dot{y}_2 + k_2 y_2 - c_1 \dot{y}_1 = u_2$$

state space variables

$$x_1 = y_1 \quad x_2 = \dot{y}_1 \quad x_3 = y_2 \quad x_4 = \dot{y}_2$$

state space equation

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -k_1 m_1^{-1} & -c_1 m_1^{-1} & 0 & c_1 m_1^{-1} \\ 0 & 0 & 0 & 1 \\ 0 & c_1 m_2^{-1} & -k_2 m_2^{-1} & -c_1 m_2^{-1} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 & 0 \\ m_1^{-1} & 0 \\ 0 & 0 \\ 0 & m_2^{-1} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + 0 \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} \quad (\text{i.e. } [D]=0)$$

Alternative method:

The two equations of motion can be rewritten in matrix form as follows:

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \end{Bmatrix} + \begin{bmatrix} c_1 & -c_1 \\ -c_1 & c_2 \end{bmatrix} \begin{Bmatrix} \dot{y}_1 \\ \dot{y}_2 \end{Bmatrix} + \begin{bmatrix} k_1 & 0 \\ 0 & k_2 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix}$$

Now define the following matrices:

$$[M] = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}, \quad [\bar{C}] = \begin{bmatrix} c_1 & -c_1 \\ -c_1 & c_2 \end{bmatrix}, \quad \text{and } [K] = \begin{bmatrix} k_1 & 0 \\ 0 & k_2 \end{bmatrix}$$

One can represent the state matrices as following:

$$[A] = \begin{bmatrix} [I] & [0] \\ -[M]^{-1}[\bar{C}] & -[M]^{-1}[K] \end{bmatrix}, \quad [B] = \begin{bmatrix} [0] \\ -[M]^{-1} \end{bmatrix}, \quad [C] = \begin{bmatrix} [0] & [I] \end{bmatrix}, \quad \text{and } [D]=0$$

$$[M]^{-1}=\frac{1}{m_1m_2}\begin{bmatrix}m_2&0\\0&m_1\end{bmatrix}=\begin{bmatrix}m_1^{-1}&0\\0&m_2^{-1}\end{bmatrix}$$

$$[M]^{-1}[\bar{C}]=\begin{bmatrix}m_1^{-1}&0\\0&m_2^{-1}\end{bmatrix}\begin{bmatrix}c_1&-c_1\\-c_1&c_2\end{bmatrix}=\begin{bmatrix}c_1m_1^{-1}&-c_1m_1^{-1}\\-c_1m_2^{-1}&c_2m_2^{-1}\end{bmatrix}$$

$$[M]^{-1}[K]=\begin{bmatrix}m_1^{-1}&0\\0&m_2^{-1}\end{bmatrix}\begin{bmatrix}k_1&0\\0&k_2\end{bmatrix}=\begin{bmatrix}k_1m_1^{-1}&0\\0&k_2m_2^{-1}\end{bmatrix}$$

$$[A]=\begin{bmatrix}1&0&0&0\\0&1&0&0\\-c_1m_1^{-1}&c_1m_1^{-1}&-k_1m_1^{-1}&0\\c_1m_2^{-1}&-c_2m_2^{-1}&0&-k_2m_2^{-1}\end{bmatrix},$$

$$[B]=\begin{bmatrix}0&0\\0&0\\m_1^{-1}&0\\0&m_2^{-1}\end{bmatrix},[C]=\begin{bmatrix}0&0&1&0\\0&0&0&1\end{bmatrix}$$

$$[D]=0$$

4. Referring to the system shown in Figure (3), determine the values of K_1 and K_2 such that maximum overshoot will be 4.6 % and settling time will be 1.5 second (2% percent criteria).

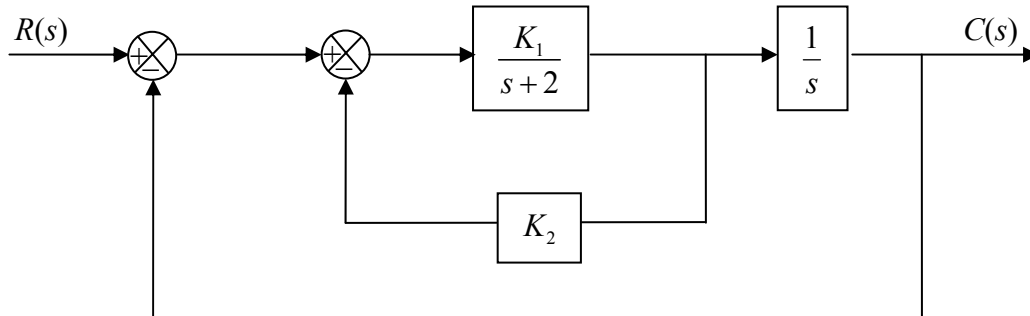


Figure (3)

Solution:

$$\frac{C(s)}{R(s)} = \frac{K_1}{s^2 + (2 + K_1 K_2)s + K_1}$$

$$M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} = 0.046 \Rightarrow \zeta = 0.7$$

$$t_s = \frac{4}{\zeta\omega_n} = 1.5 \Rightarrow \omega_n = 3.8$$

$$K_1 = \omega_n^2 = 3.8^2 = 14.51$$

$$2 + K_1 K_2 = 2\zeta\omega_n = 2(0.7)(3.8) = 6.46 \Rightarrow K_2 = 0.3$$

5. Consider the following equation of motion of a mechanical system:

$$\ddot{y} + 5\dot{y} + 6y = \dot{u} + 6u$$

a) Determine the transfer function $Y(s)/U(s)$ of the system.

Solution:

Take Lablace for both sides:

$$Ys^2 + 5Ys + 6Y = Us + 6U$$

$$\frac{Y(s)}{U(s)} = \frac{s + 6}{s^2 + 5s + 6}$$

b) Obtain the state space representation of the system in controllable canonical form.

Solution:

$$\ddot{y} + 5\dot{y} + 6y = 0 \quad \ddot{u} + \dot{u} + 6u$$

$$\ddot{y} + a_1\dot{y} + a_2y = b_0\ddot{u} + b_1\dot{u} + b_2u$$

Controllable canonical form

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ -6 & -5 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 6 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + 0u$$

c) Is the system completely state controllable? Why?

Solution:

Controllability matrix:

$$[B \quad AB] = \begin{bmatrix} 0 & 0 \\ 1 & 5 \end{bmatrix}$$

Since the controllability matrix is of rank 1, then the system is not completely state controllable.

d) Is the system completely state observable? Why?

Solution:

Observability matrix:

$$[C^* \quad A^*C^*] = \begin{bmatrix} 6 & 0 \\ 1 & -5 \end{bmatrix}$$

Since the observability matrix is of rank 2, then the system is completely state observable.