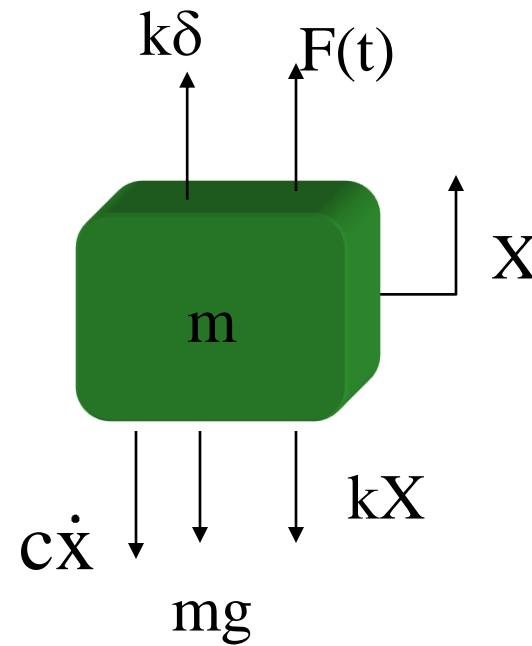
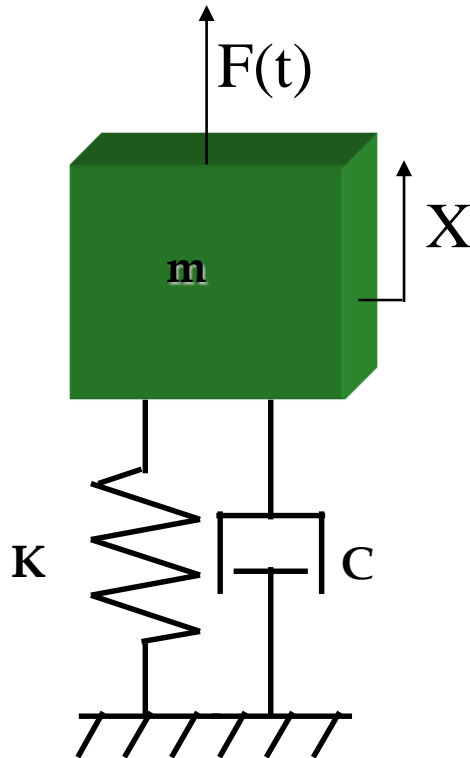


Single Degree of Freedom Forced Vibration

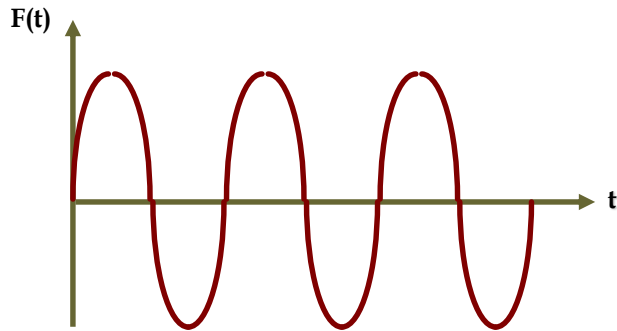
Free Body Diagram



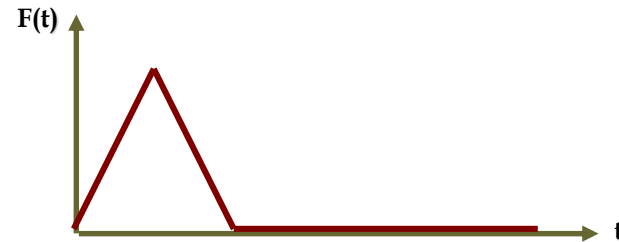
$$m\ddot{X} + c\dot{X} + kX = F(t)$$

There are 4 categories of $F(t)$:

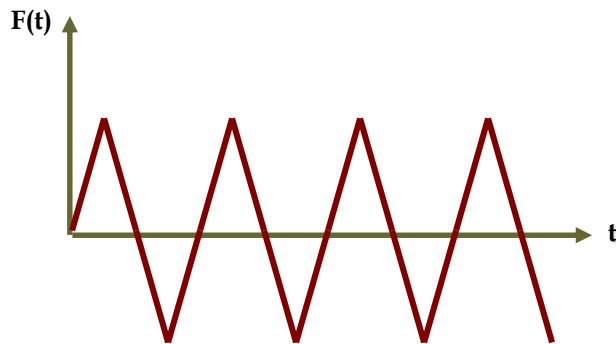
1) Harmonic (sin, cos)



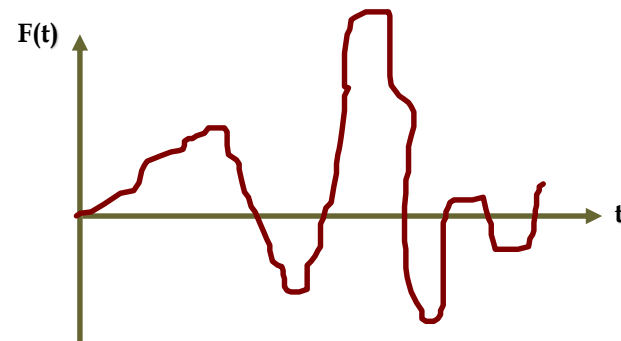
3) Transient



2) Periodic



4) Random

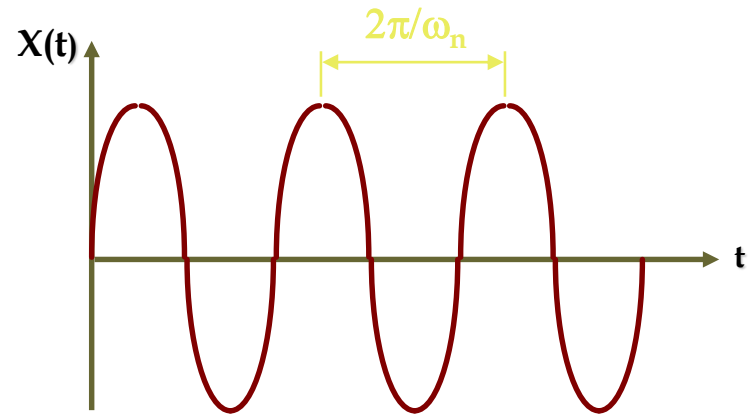


If $F(t) = F_0 \cos(\omega t)$ $\Rightarrow m\ddot{X} + c\dot{X} + kX = F_0 \cos(\omega t)$

If $c = 0$ $\Rightarrow m\ddot{X} + kX = F_0 \cos(\omega t)$

$$X(t) = X_h(t) + X_p(t)$$

$$X(t) = X_{\text{trans}}(t) + X_{\text{ss}}(t)$$

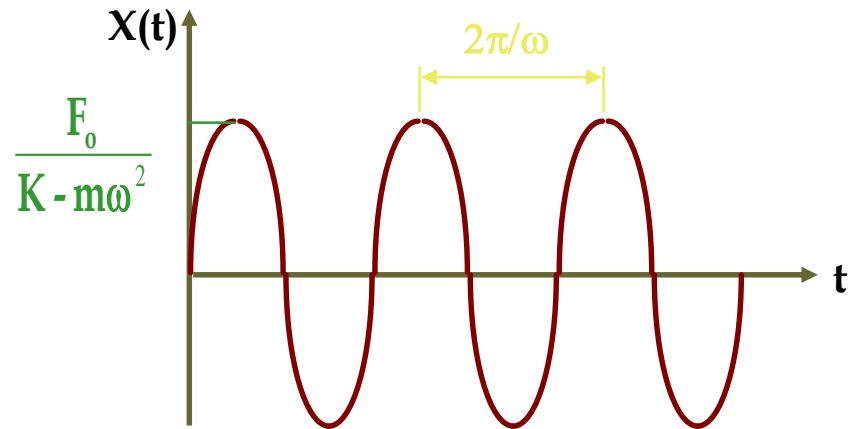


$$X_{\text{trans}}(t) = A \cos(\omega_n t) + B \sin(\omega_n t)$$

$$X_{ss}(t) = X \cos(\omega t)$$

$$\dot{X}_{ss}(t) = -\omega X \sin(\omega t)$$

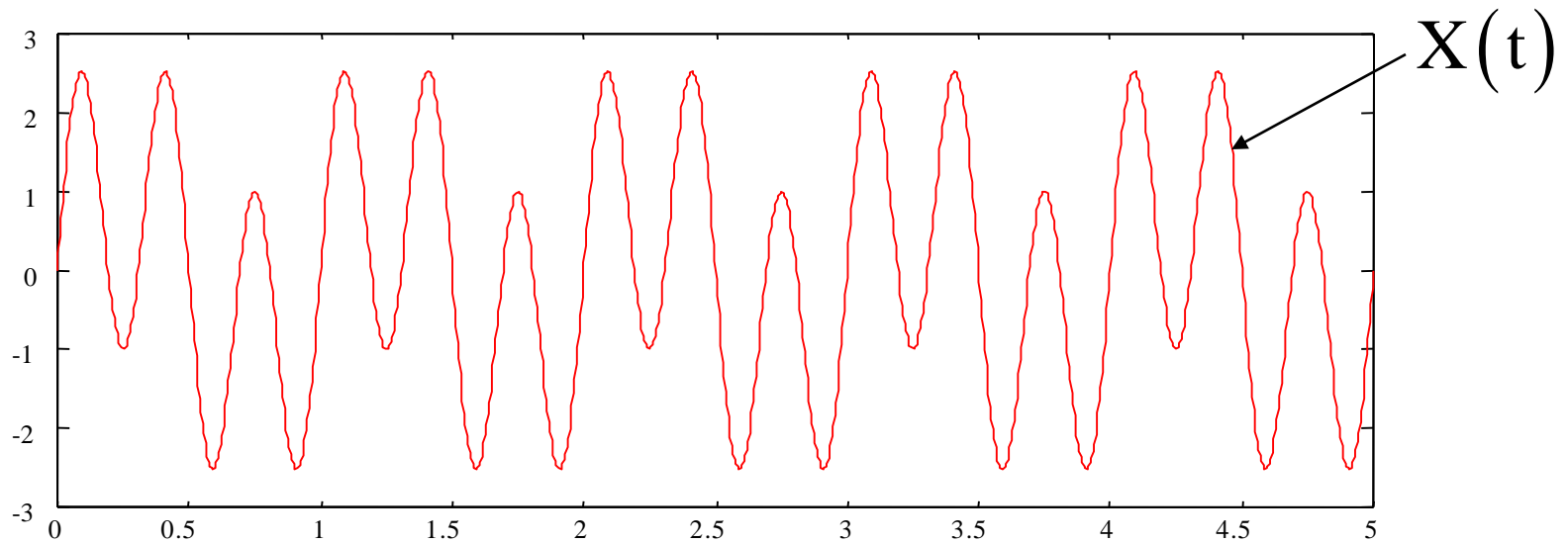
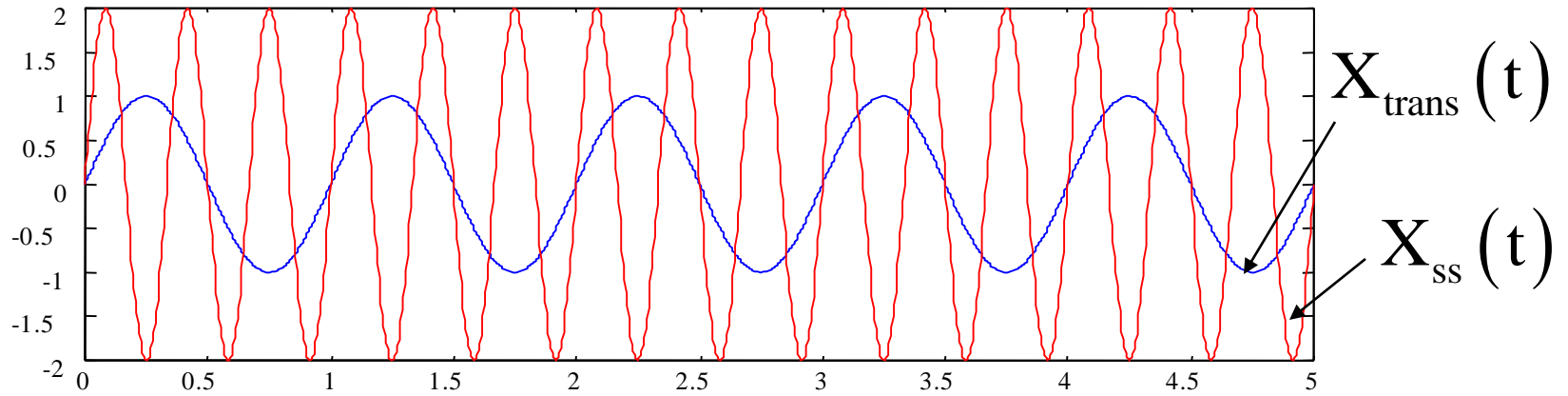
$$\ddot{X}_{ss}(t) = -\omega^2 X \cos(\omega t)$$



$$-m\omega^2 X \cos(\omega t) + kX \cos(\omega t) = F_0 \cos(\omega t)$$

$$X = \frac{F_0}{k - m\omega^2}$$

$$X_{ss}(t) = \frac{F_0}{k - m\omega^2} \cos(\omega t)$$



$$X(t) = A \cos(\omega_n t) + B \sin(\omega_n t) + \frac{F_0}{k - m\omega^2} \cos(\omega t)$$

$$X(0) = A + \frac{F_0}{K - m\omega^2} \Rightarrow A = X(0) - \frac{F_0}{K - m\omega^2}$$

$$\dot{X}(t) = -A\omega_n \sin(\omega_n t) + B\omega_n \cos(\omega_n t) - X\omega \sin(\omega t)$$

$$\dot{X}(0) = \omega_n B \Rightarrow B = \frac{\dot{X}(0)}{\omega_n}$$

$$X(t) = \left(X(0) - \frac{F_0}{k - m\omega^2} \right) \cos(\omega_n t) + \frac{\dot{X}(0)}{\omega_n} \sin(\omega_n t) + \frac{F_0}{k - m\omega^2} \cos(\omega t)$$

Case 1: *if $\omega \ll \omega_n$*

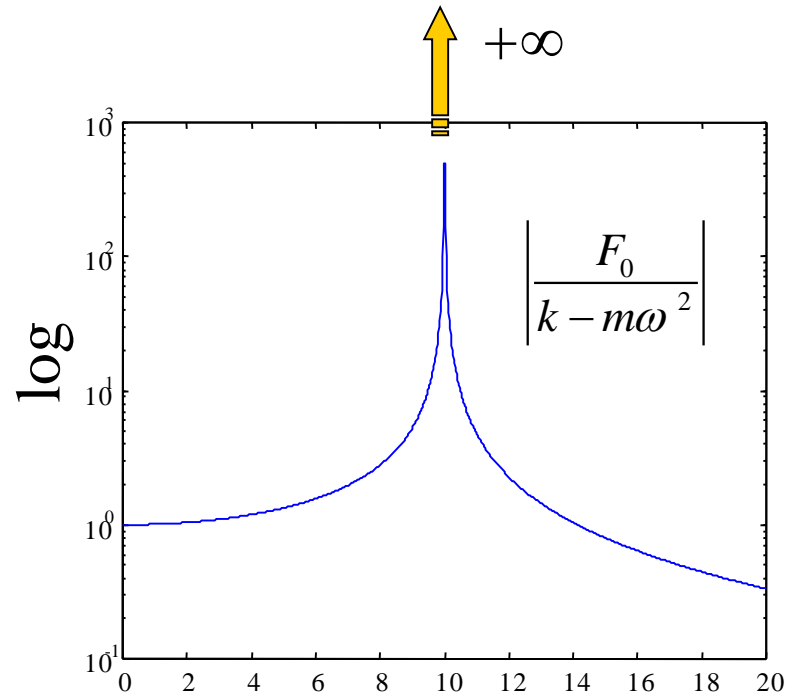
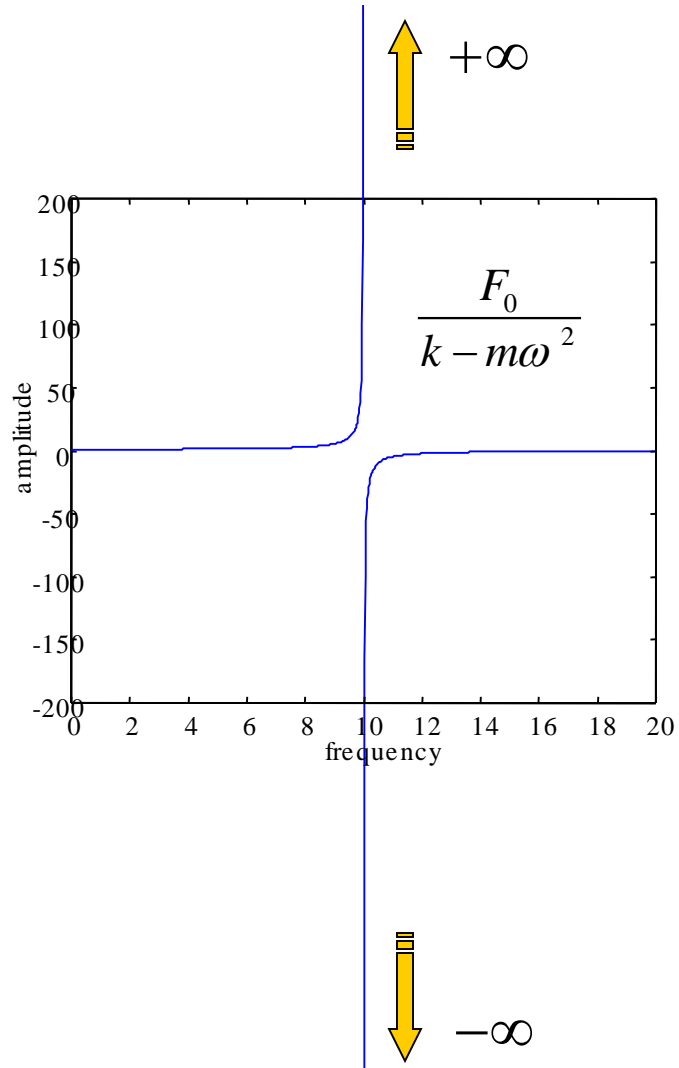
$$\frac{F_o}{k - m\omega^2} = \frac{F_o}{k - \frac{k\omega^2}{\omega_n^2}} \approx \frac{F_o}{k} = \delta_{st}$$

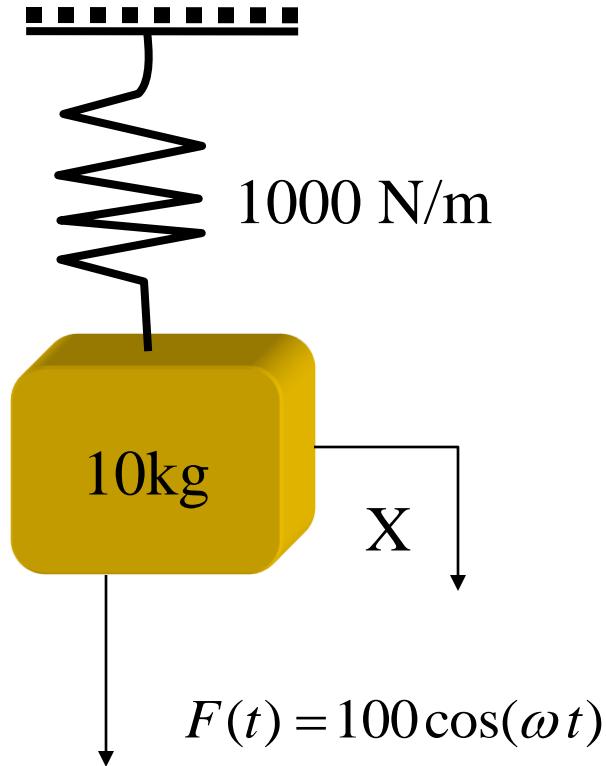
Case 2: *if $\omega \gg \omega_n$*

$$\frac{F_o}{k - m\omega^2} = \frac{F_o}{k - \frac{k\omega^2}{\omega_n^2}} \approx -\frac{F_o}{\frac{k\omega^2}{\omega_n^2}}$$

Case 3: *if $\omega = \omega_n$*

$$\frac{F_o}{k - m\omega^2} = \frac{F_o}{0} \approx \infty \quad \Rightarrow \quad X(t) = \frac{F_o \omega_n t}{2k} \sin(\omega_n t)$$



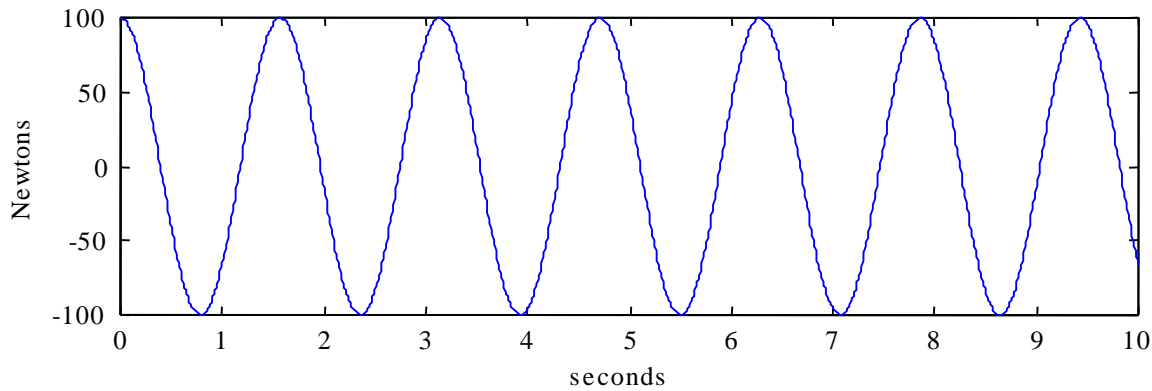


Steady State Response

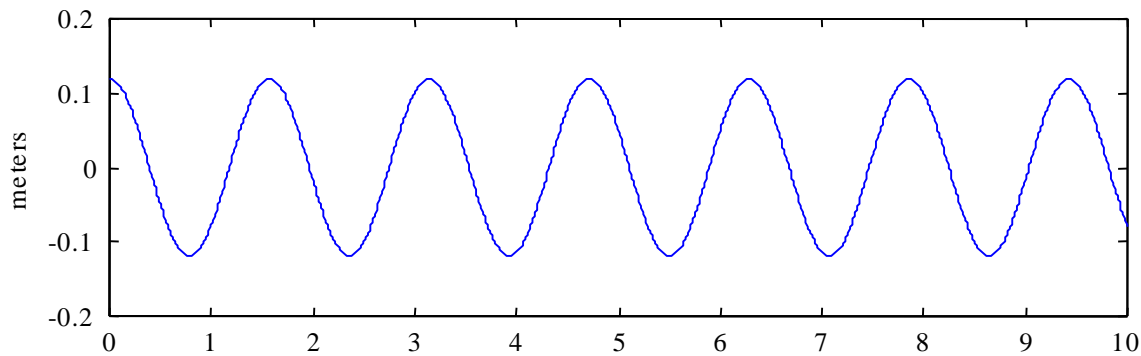
$$x(t) = \frac{100}{1000 - 10\omega^2} \cos(\omega t)$$

$$\omega_n = \sqrt{\frac{1000}{10}} = 10 \text{ rad/sec}$$

Case 1: *if* $\omega \ll \omega_n$

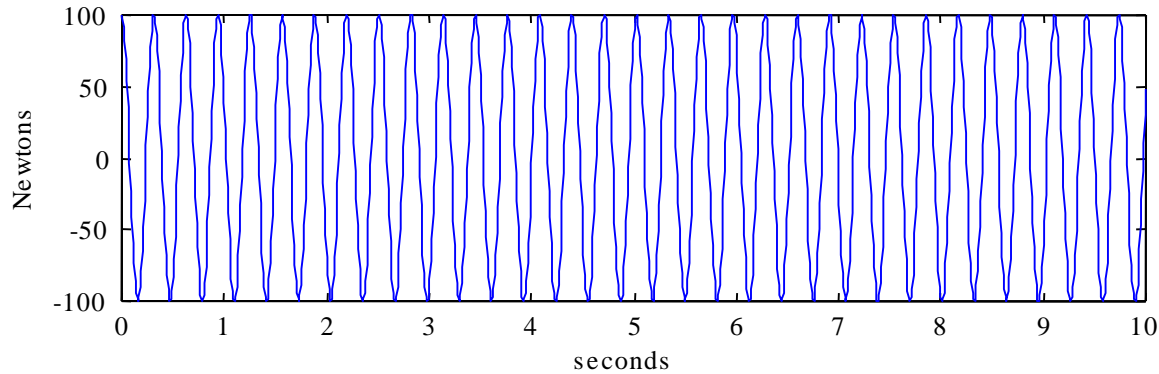


$$F(t) = 100 \cos(4t)$$

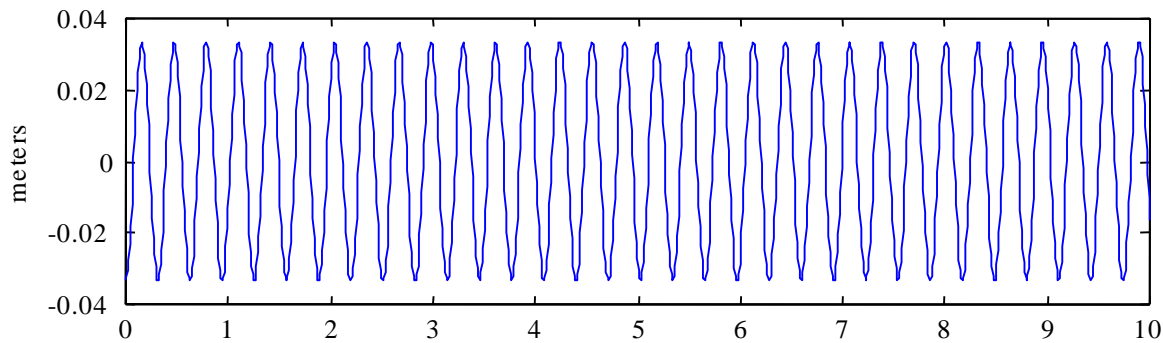


$$X(t) = \frac{100}{840} \cos(4t)$$

Case 2: $\omega \gg \omega_n$

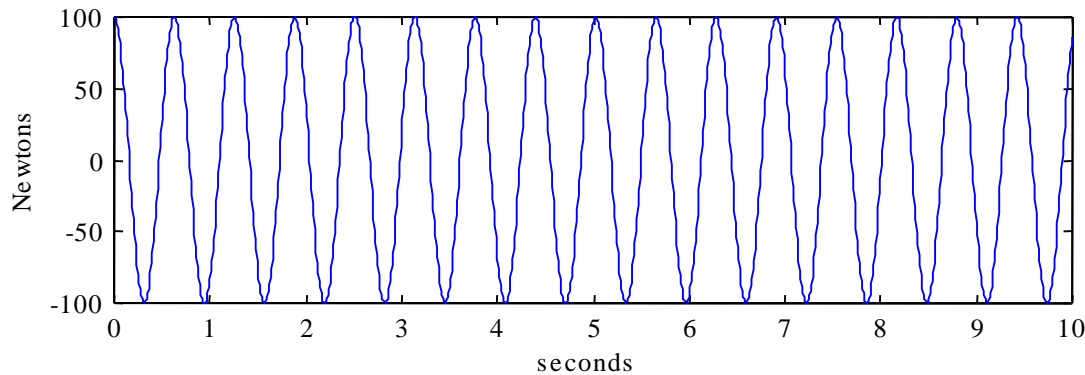


$$F(t) = 100 \cos(20t)$$

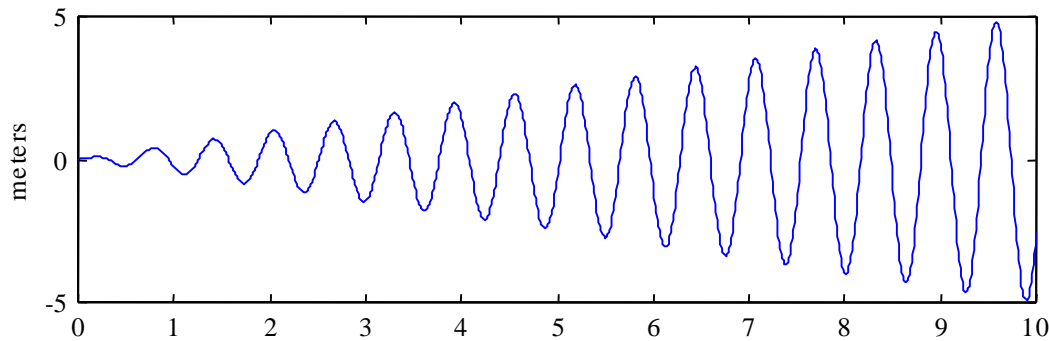


$$X(t) = -\frac{100}{3000} \cos(20t)$$

Case 3: *if $\omega = \omega_n$*



$$F(t) = 100 \cos(10t)$$



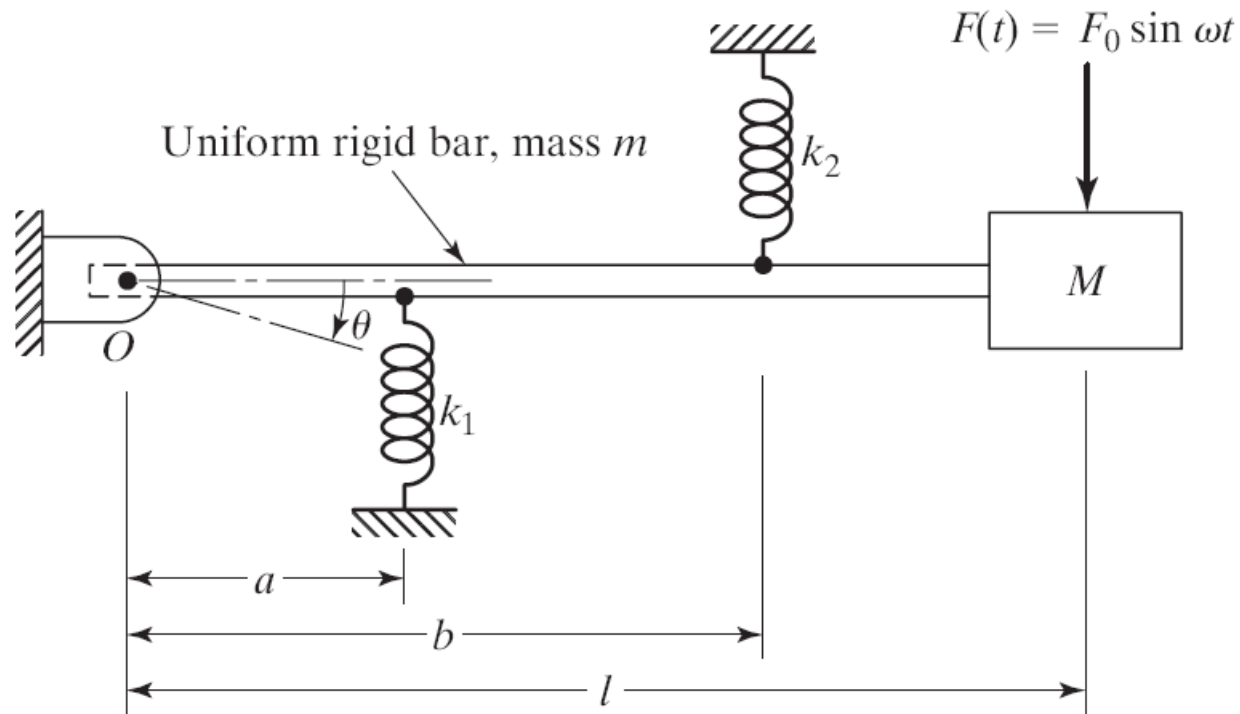
$$X(t) = \frac{1000t}{2000} \sin(10t)$$

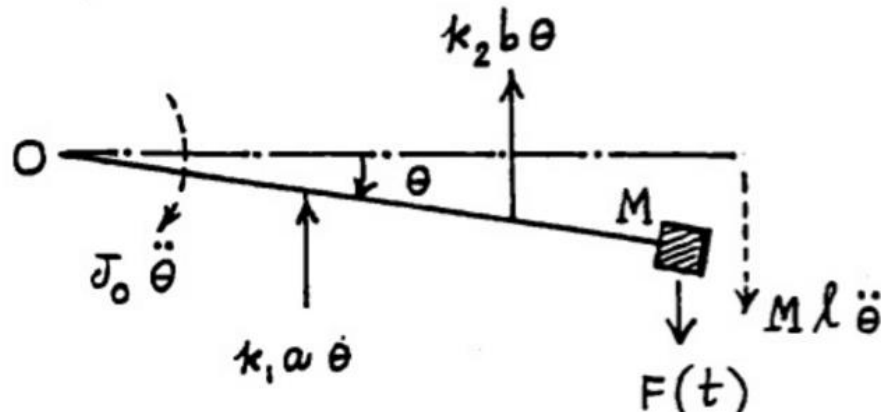
Example1:

A spring-mass system, with a spring stiffness of 5,000 N/m, is subjected to a harmonic force of magnitude 30 N and frequency 20 Hz. The mass is found to vibrate with an amplitude of 0.2 m. Assuming that vibration starts from rest ($x_0 = \dot{x}_0 = 0$), determine the mass of the system.

Example2:

Derive the equation of motion and find the steady-state response of the system shown in Fig. 3.44 for rotational motion about the hinge O for the following data: $k_1 = k_2 = 5000$ N/m, $a = 0.25$ m, $b = 0.5$ m, $l = 1$ m, $M = 50$ kg, $m = 10$ kg, $F_0 = 500$ N, $\omega = 1000$ rpm.





Equation of motion for rotational motion about the hinge O:

$$(J_0 + M \ell^2) \ddot{\theta} + (k_1 a^2 + k_2 b^2) \theta = F(t) \ell = F_0 \ell \sin \omega t \quad (1)$$

Steady state response (using Eqs. (3.3) and (3.6)):

$$\theta_p(t) = \Theta \sin \omega t \quad (2)$$

$$\text{where } \Theta = \frac{F_0 \ell}{(k_1 a^2 + k_2 b^2) - (J_0 + M \ell^2) \omega^2} \quad (3)$$

$$\text{and } J_0 = \frac{m \ell^2}{12} + m \left(\frac{\ell}{2}\right)^2 = \frac{1}{3} m \ell^2 \quad (4)$$

For given data, $J_0 = \frac{1}{3} (10) (1^2) = 3.3333 \text{ kg-m}^2$, $\omega = \frac{1000 (2 \pi)}{60} = 104.72 \text{ rad/sec}$,

$$\Theta = \frac{500 (1)}{5000 (0.25^2 + 0.5^2) - (3.3333 + 50 (1^2)) (104.72^2)} = -8.5718 (10^{-4}) \text{ rad}$$